

# REXTRUDE™ FILAMENT RECYCLER

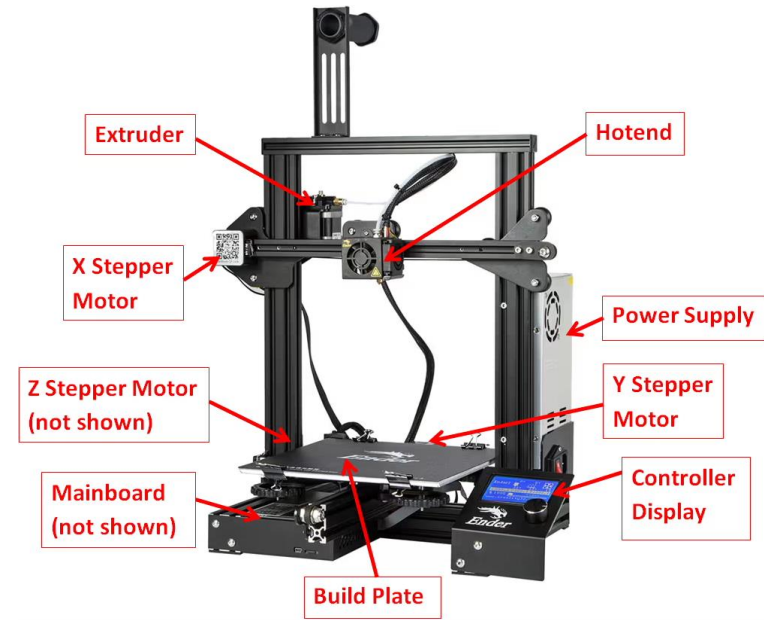
August 5, 2024  
MAE-A-221

Chase Eskridge  
Chris Lopez  
Isabella Camacho  
Gabriel Gilvary

Julia Gallo  
Nathan Berner  
Osaji Skyers  
Sophia Sayre

# Background

- 3D Printing – Additive Manufacturing Process
- Prototyping – Commonly Used
- Other uses: Functional end-use parts, Manufacturing Aids & Tools, etc.



# Project Brief

## Problem

1. Produces a lot of waste
2. Current 3D Filament Recyclers – Expensive & Slow!
3. Inaccurate Diameter Readings

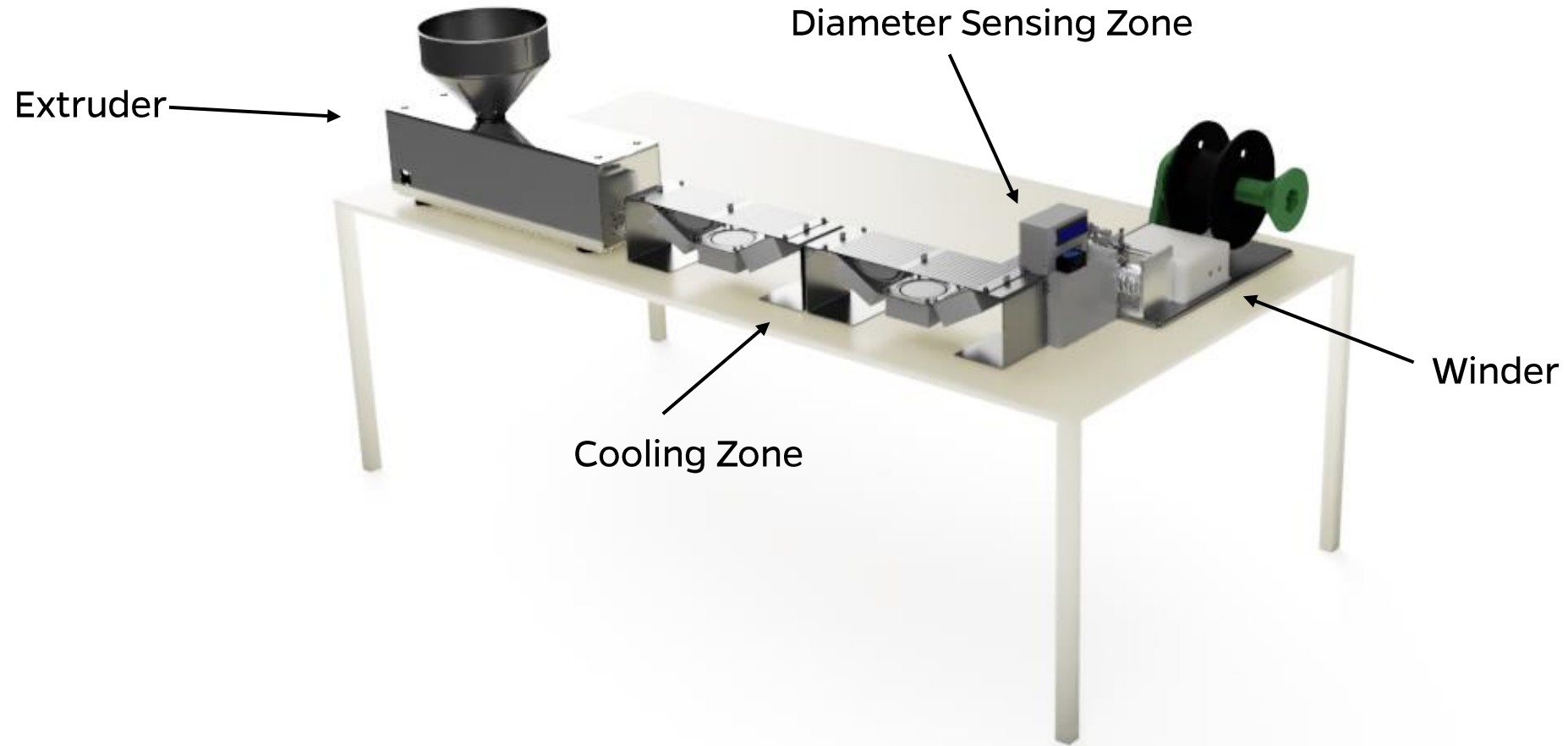
## Solution

1. Filament Recycler
2. Increased Extrusion Speeds & Affordable Costs
3. Real-time Diameter sensing zone

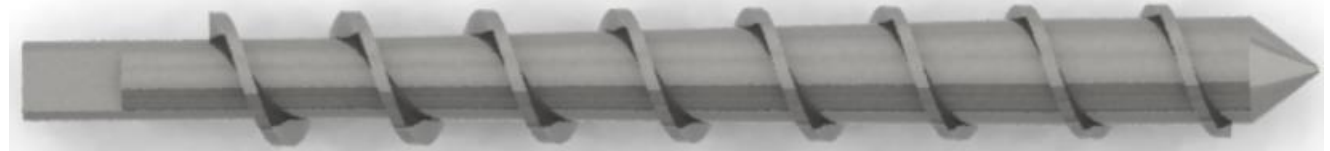
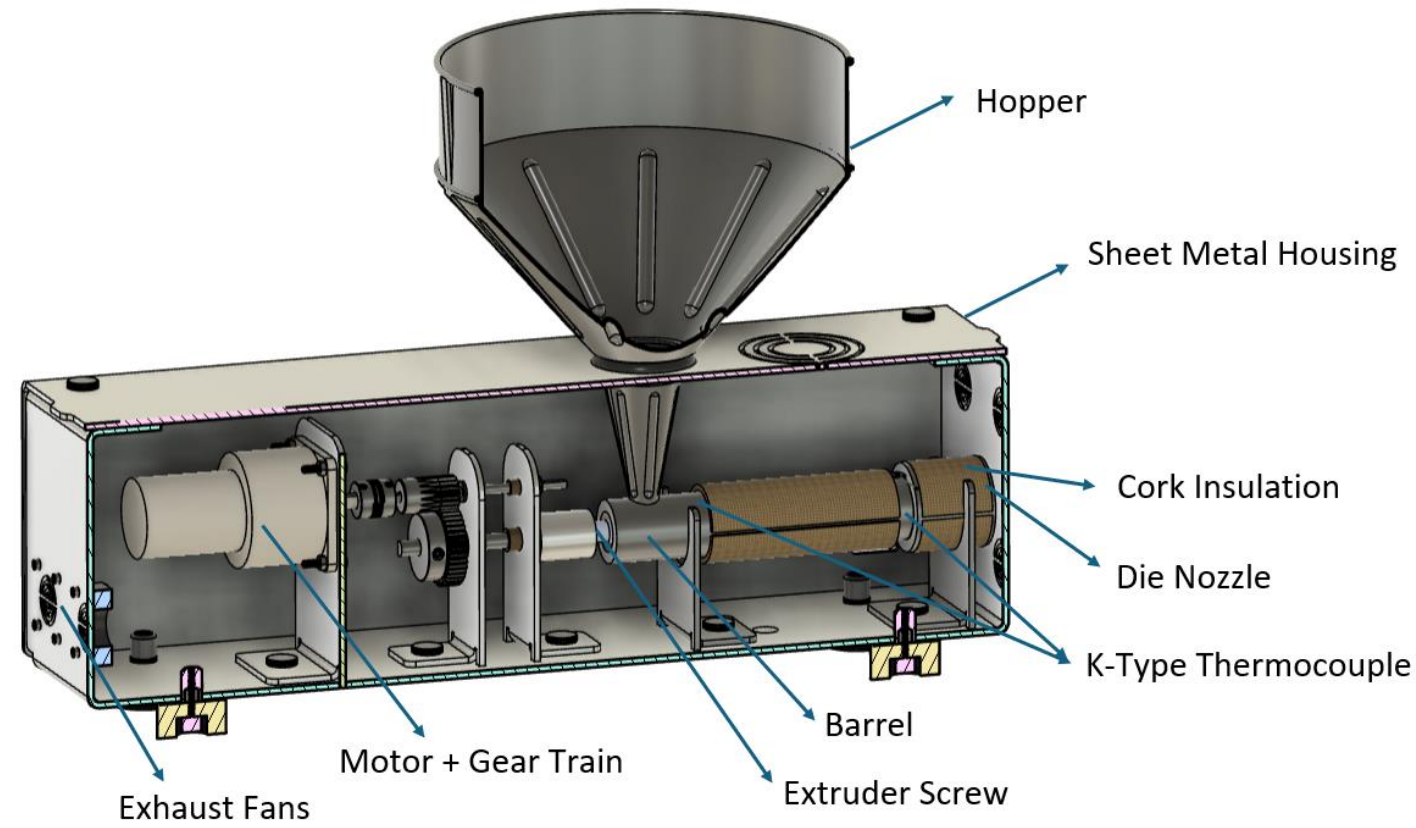


# Final Design

# Full Design Assembly



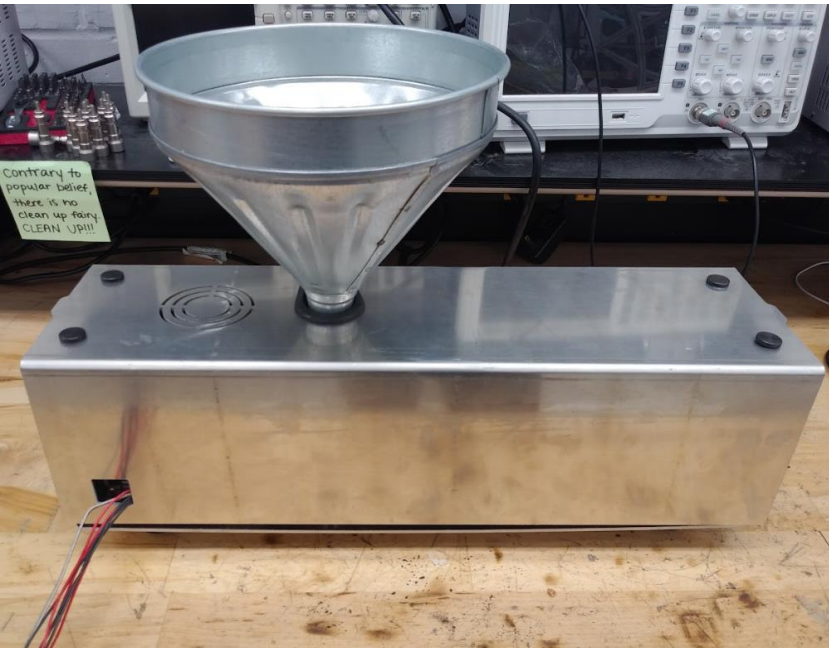
# Extruder Subsystem



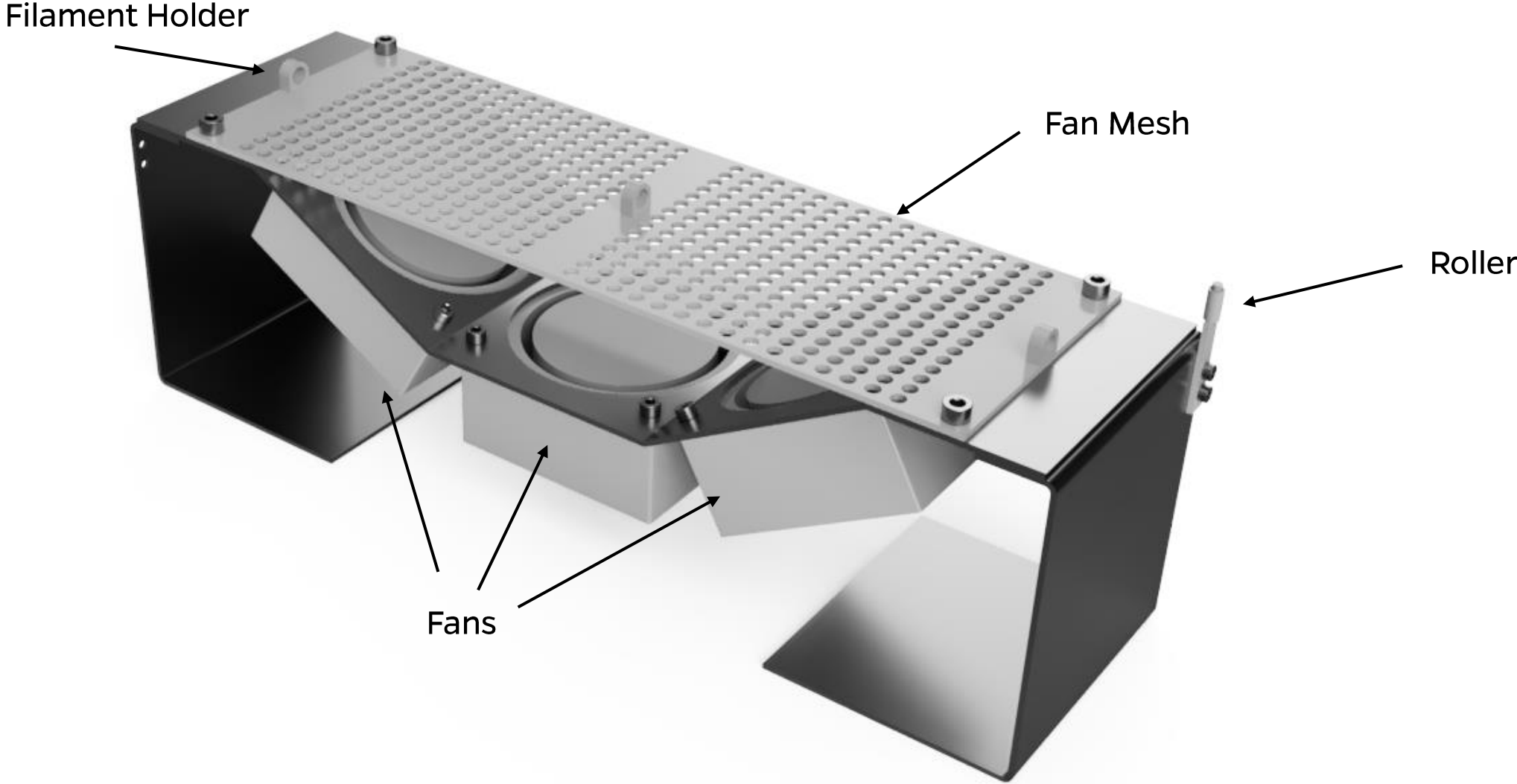
## Compressive Extruder Screw

2:1 Compression Ratio  
20.2 mm Diameter  
4 mm Channel Depth  
7.5 L/D Ratio

# Extruder Subsystem

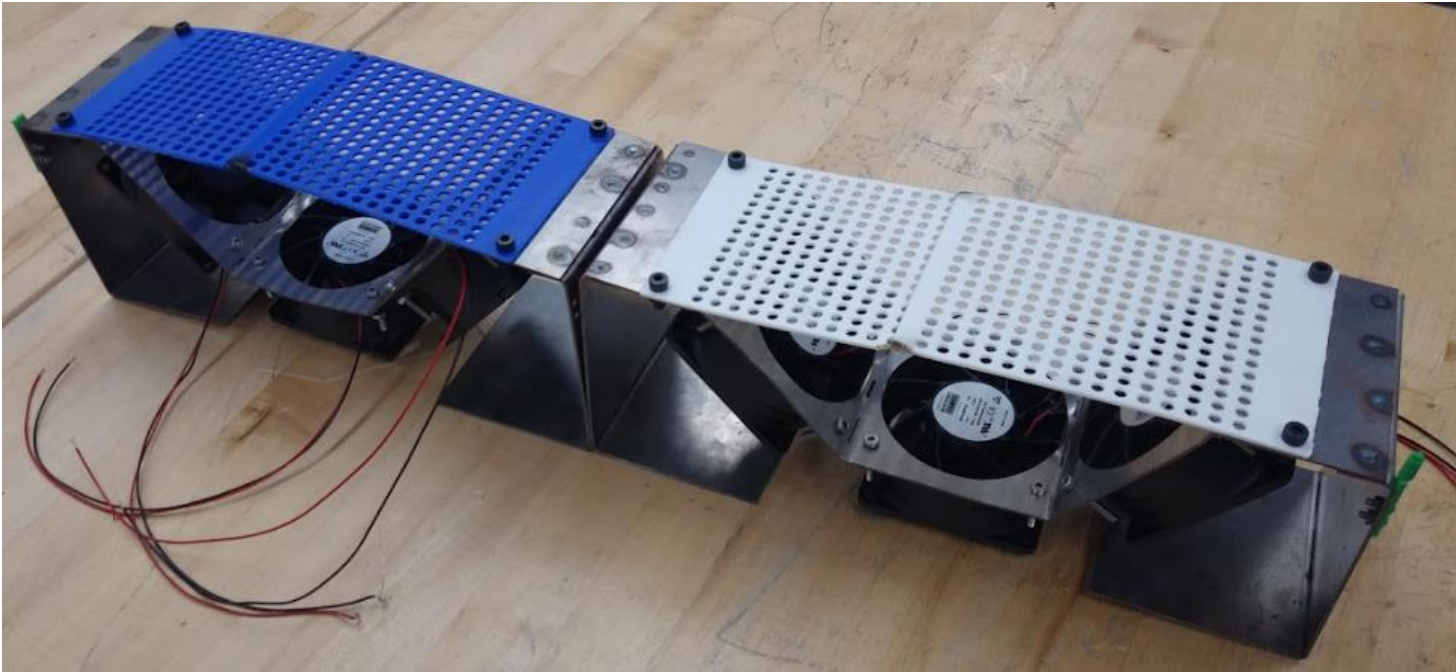


# Cooling Zone Subsystem



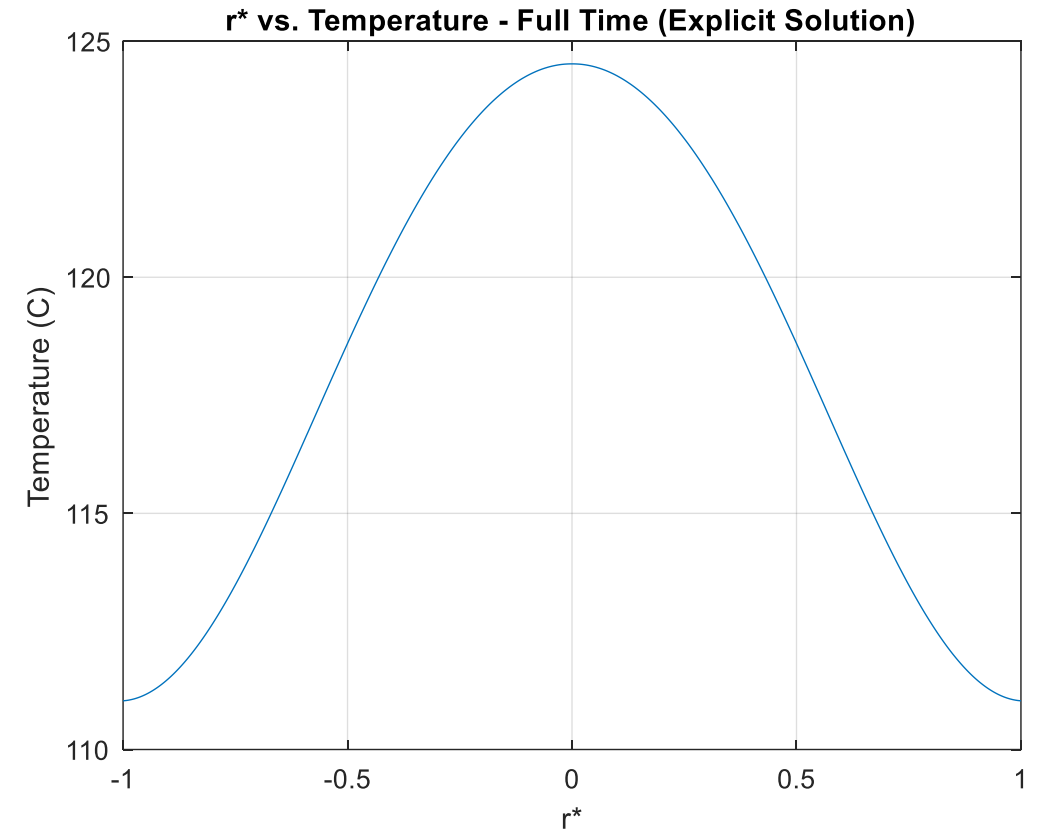
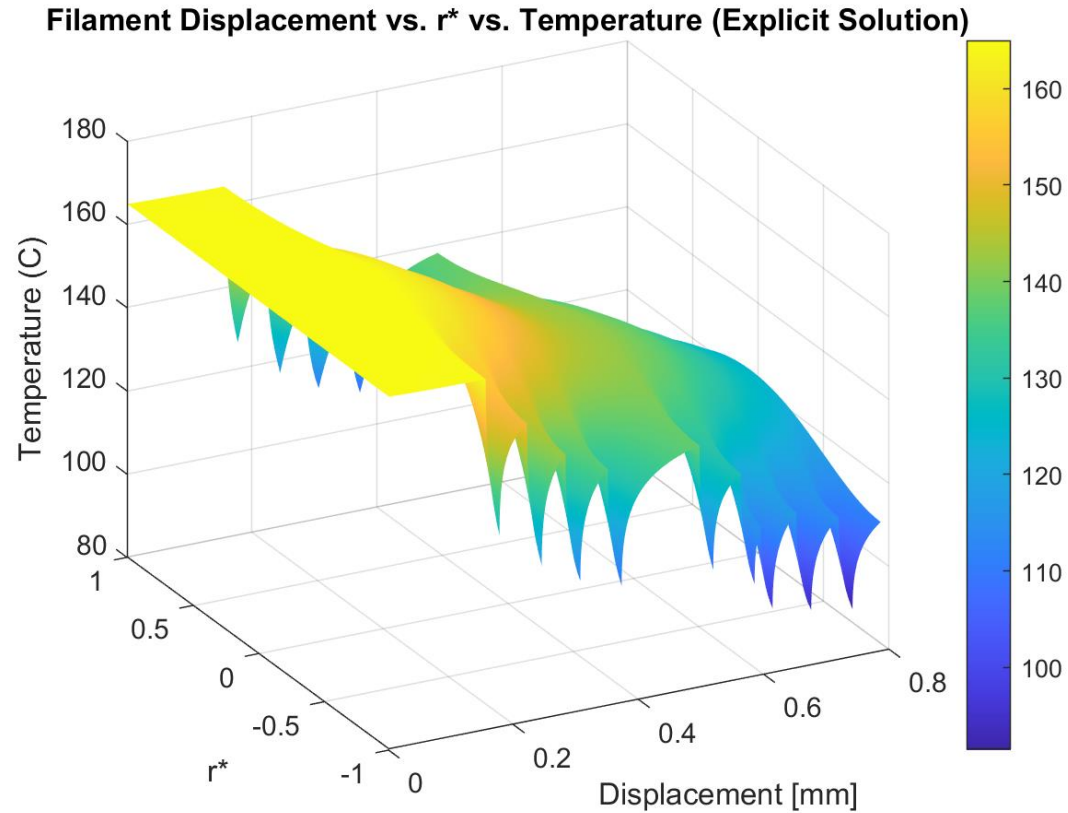


# Cooling Zone Subsystem



# Filament Cooling - Explicit Method

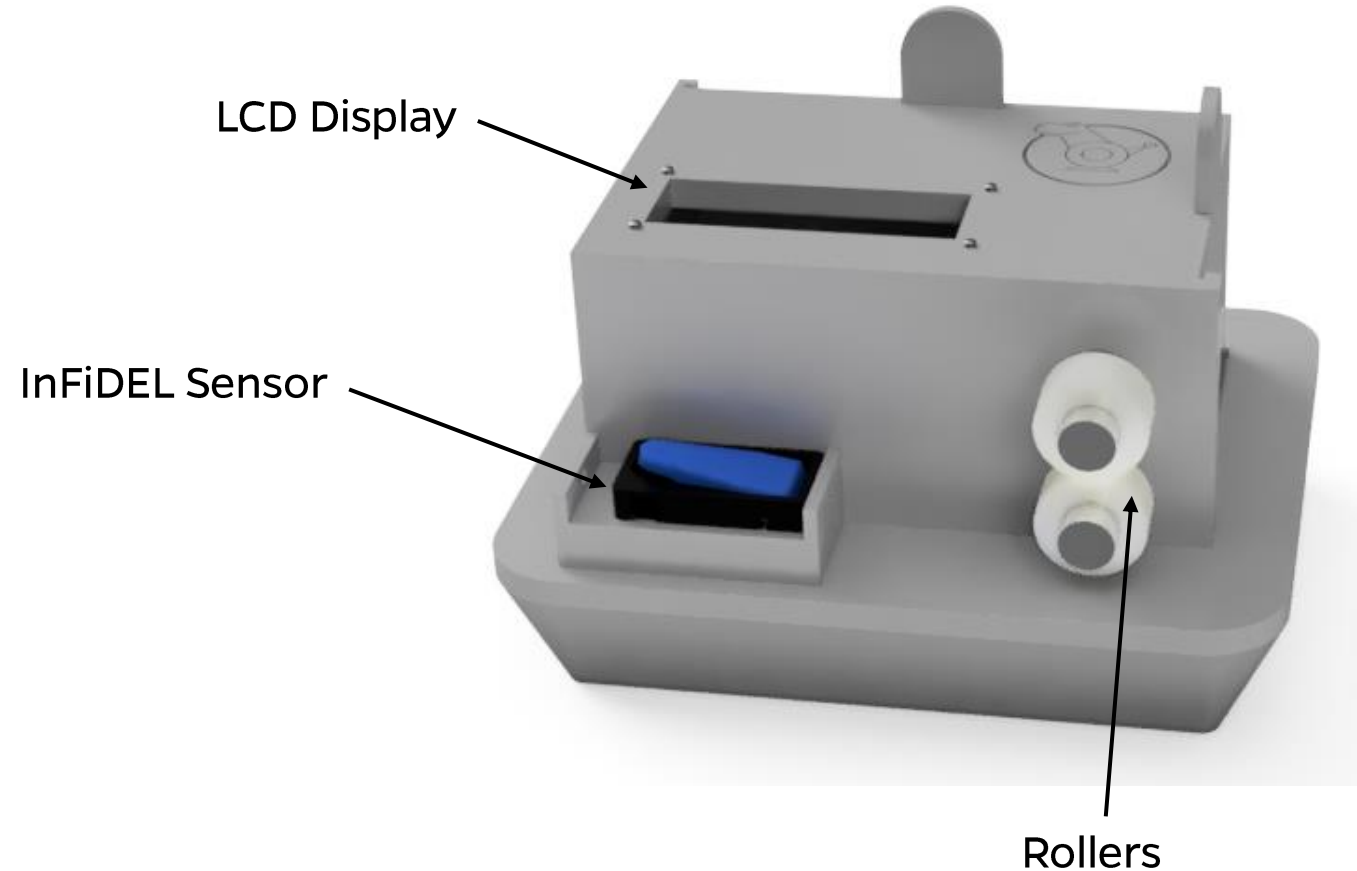
- Temperature distribution along cooling path calculated using explicit transient heat transfer:



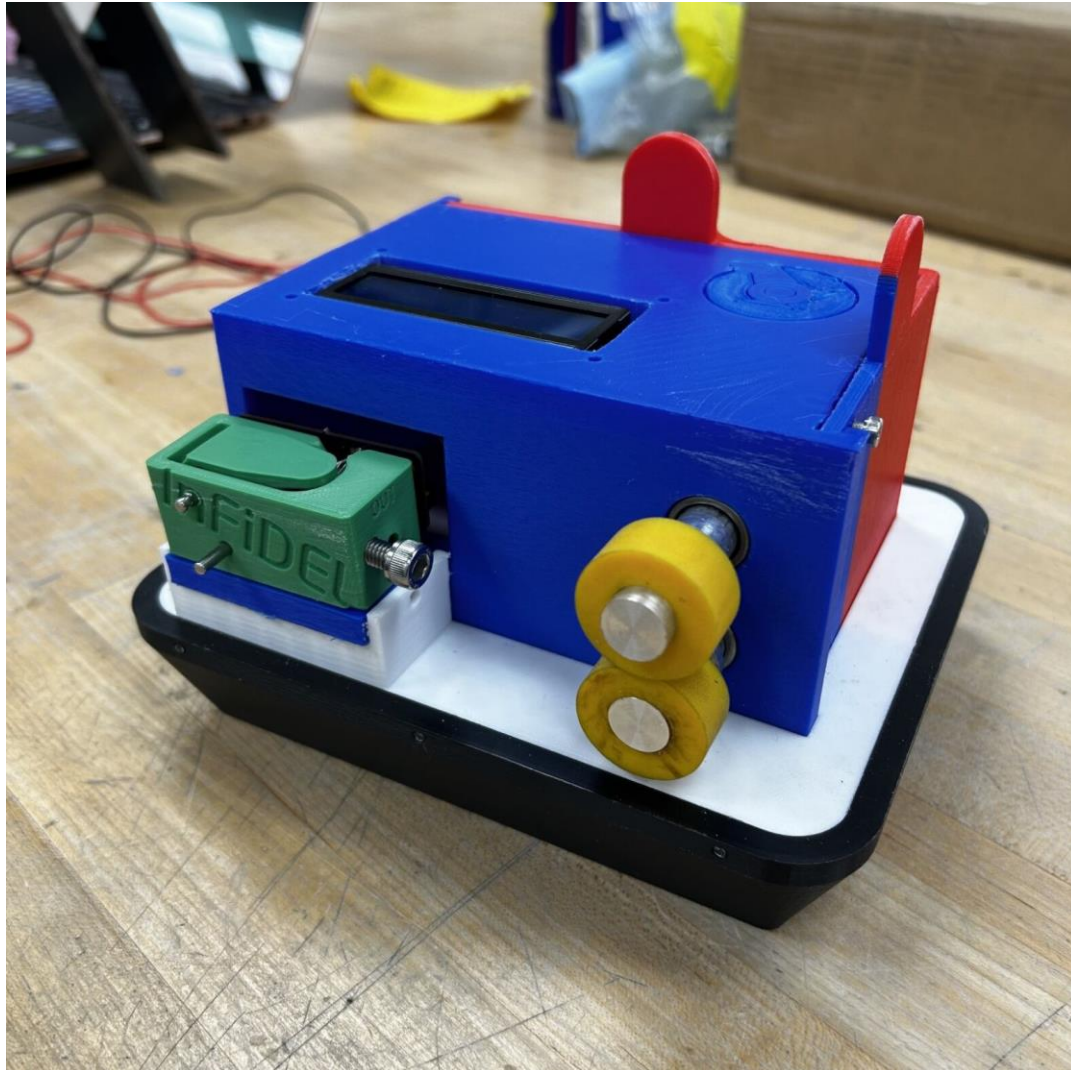
# Diameter Sensing Zone Subsystem

Uniform Deformation of extruded Filament

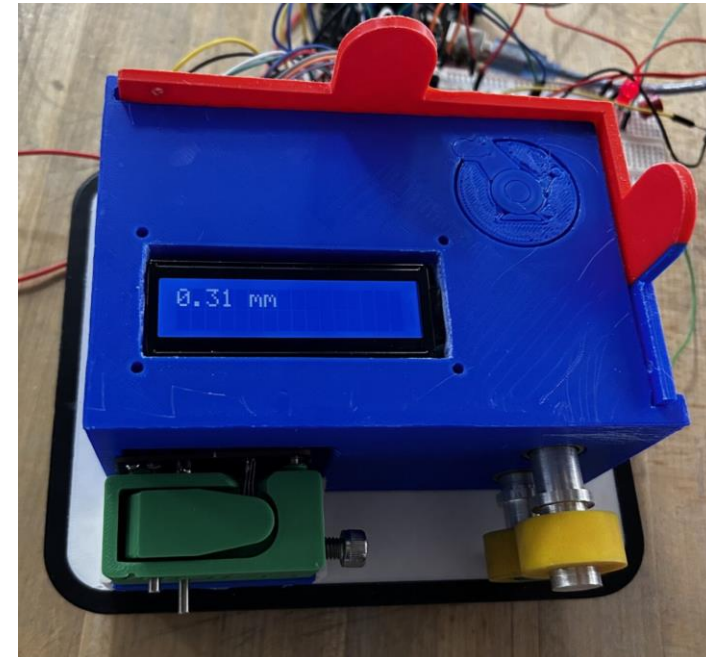
- 30-A Urethane rollers guide filament
- Driven by 12V brushless motor



# Diameter Sensing Zone Subsystem



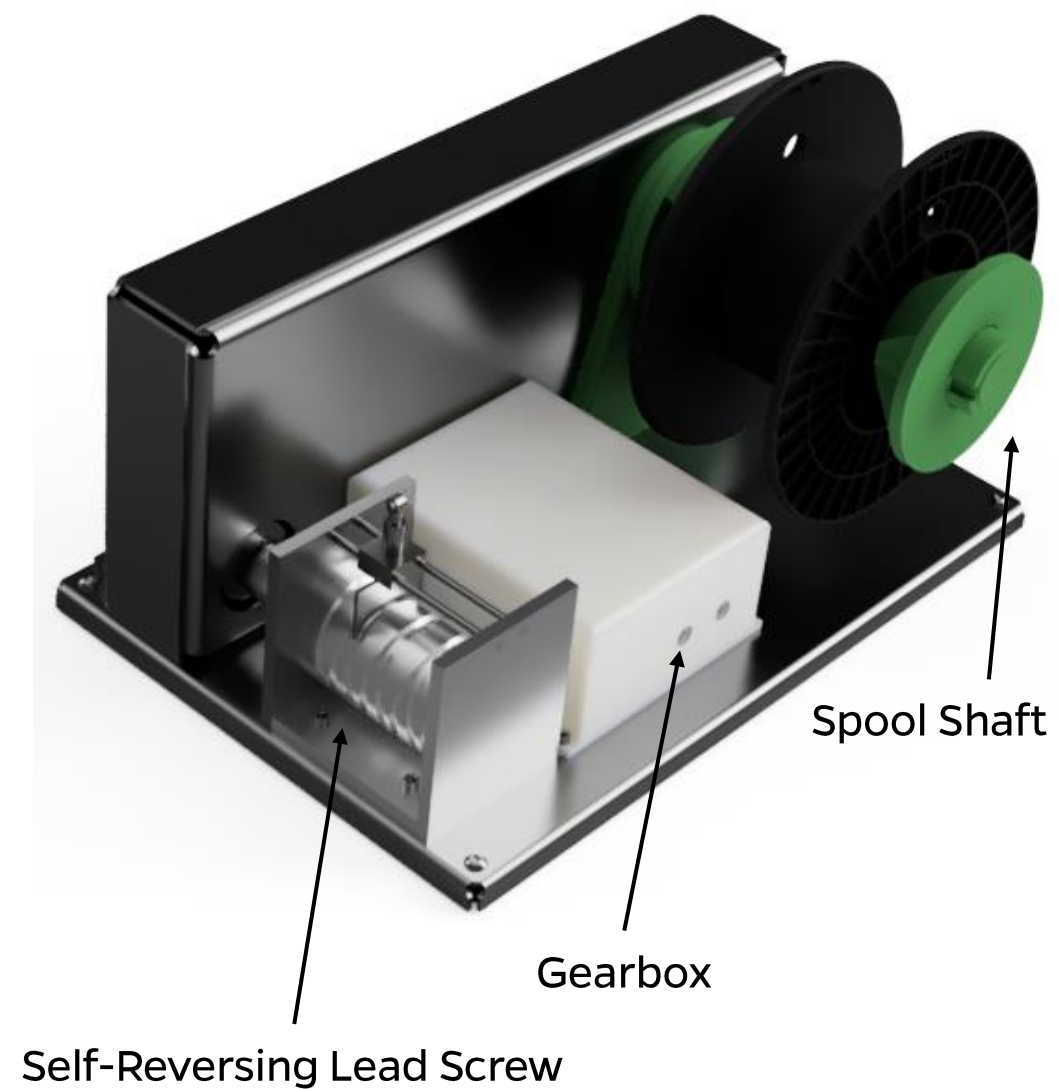
- InFiDEL sensor with hand-soldered PCB
- LCD display for readings
- 1 measurement per second
- Voltage differences measure diameter



# Winder Subsystem

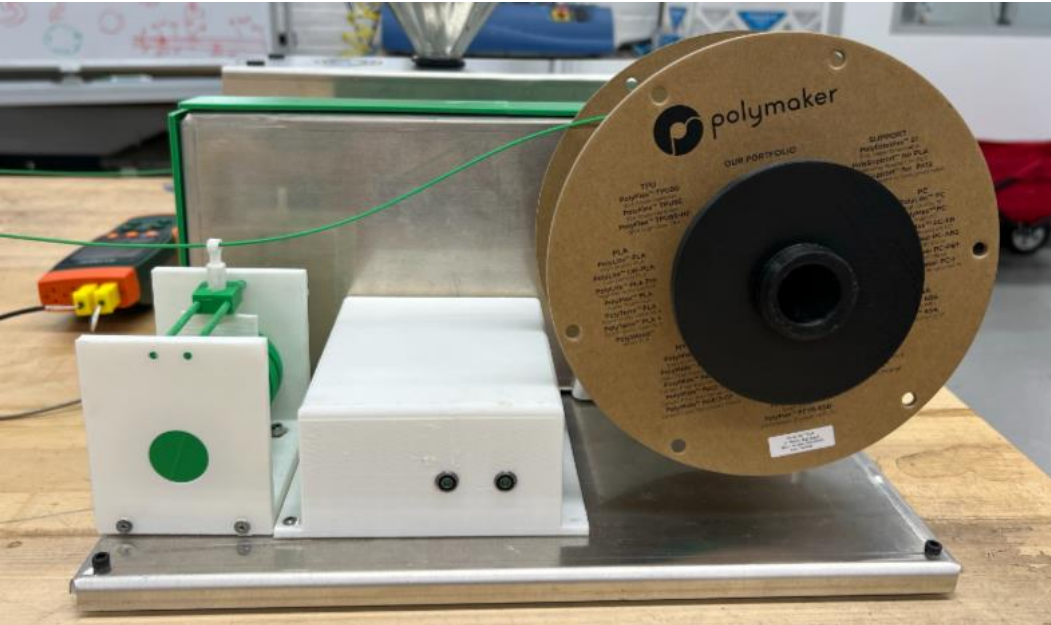
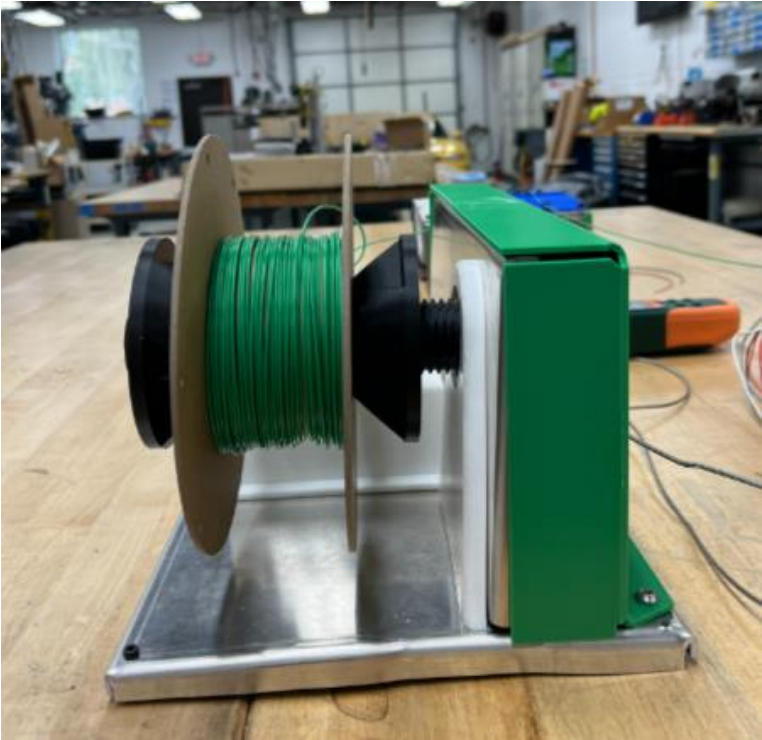
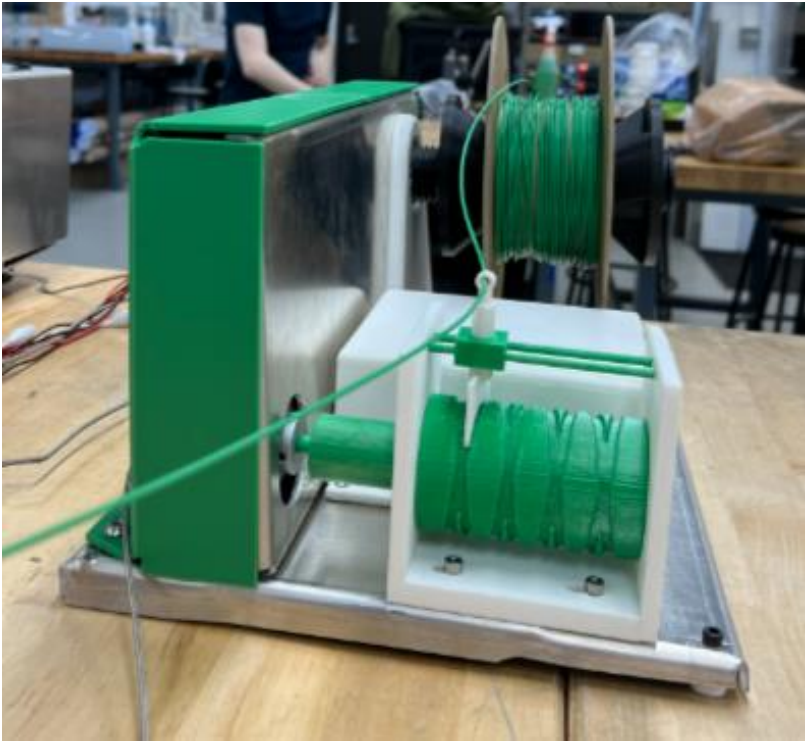
Winds filament onto a spool for 3D printing

- Aluminum housing
- Adjustable spool holder
- 3D printed gearbox
- Self-reversing screw



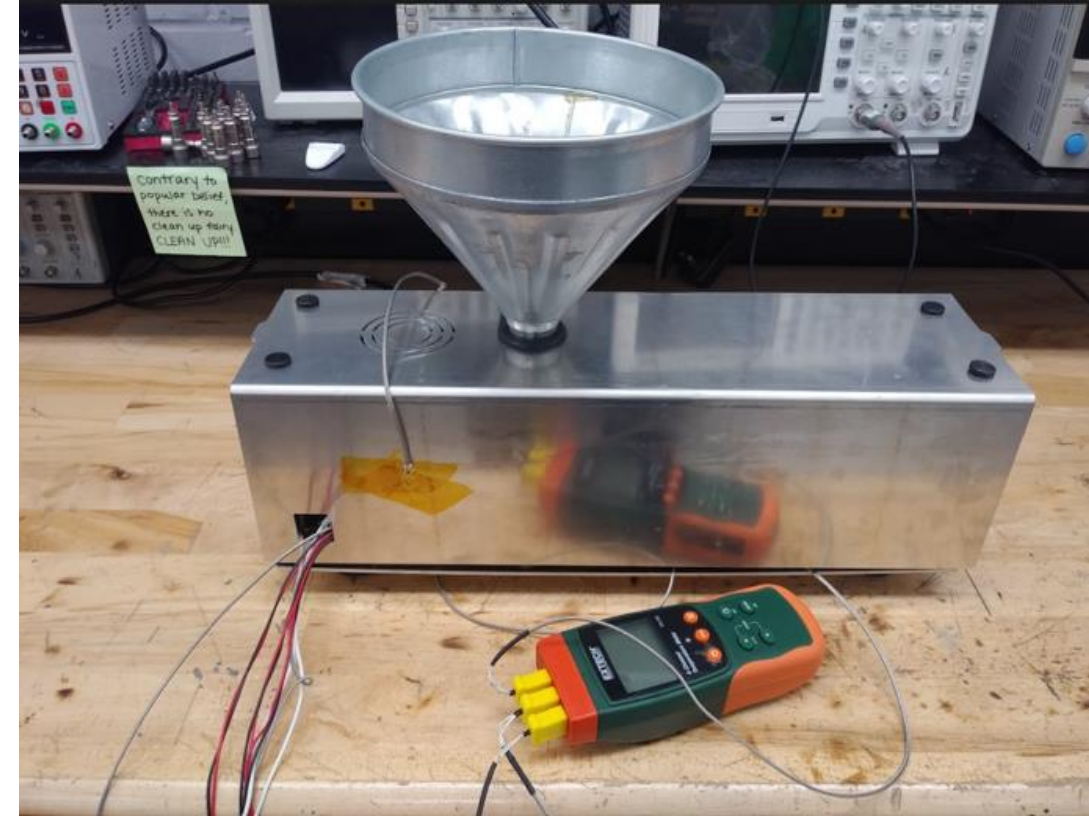
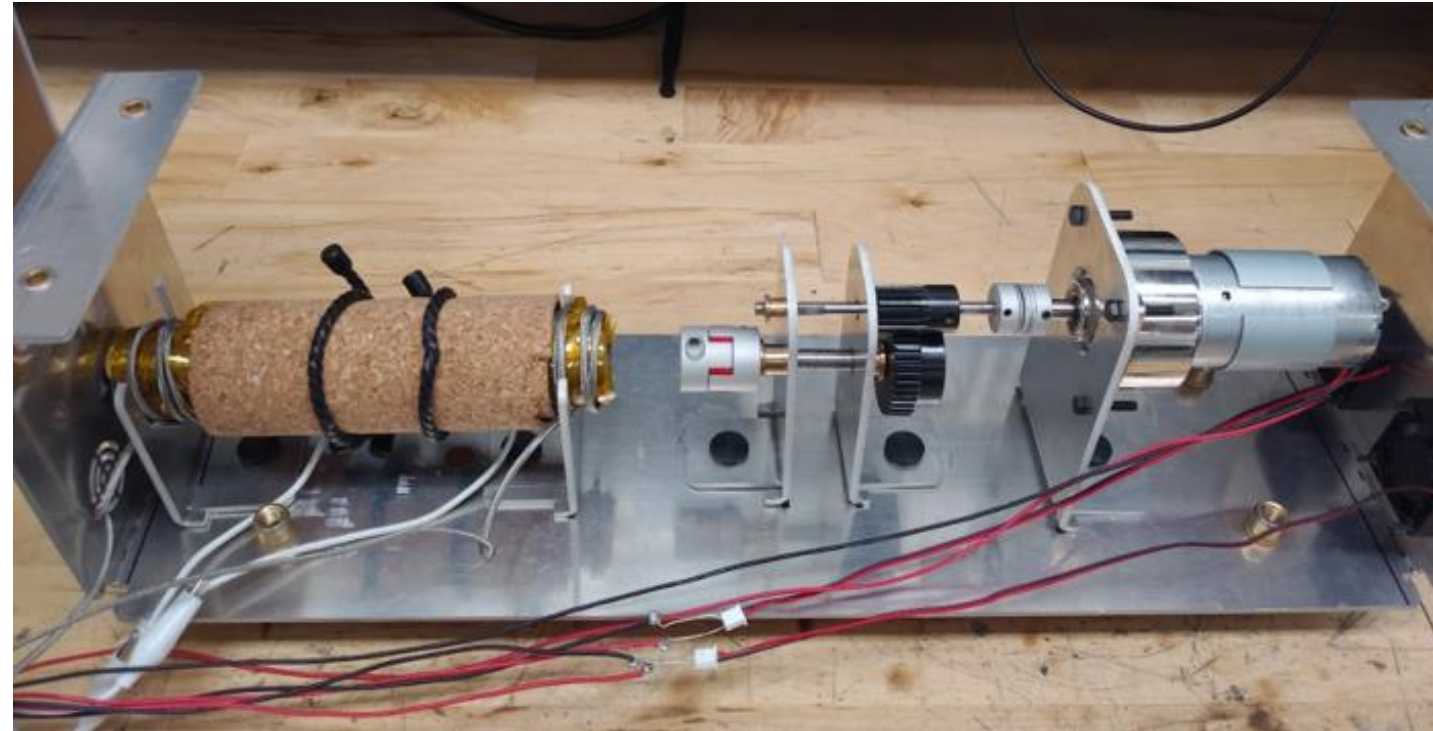
# Winder Subsystem

Prototype of the winder



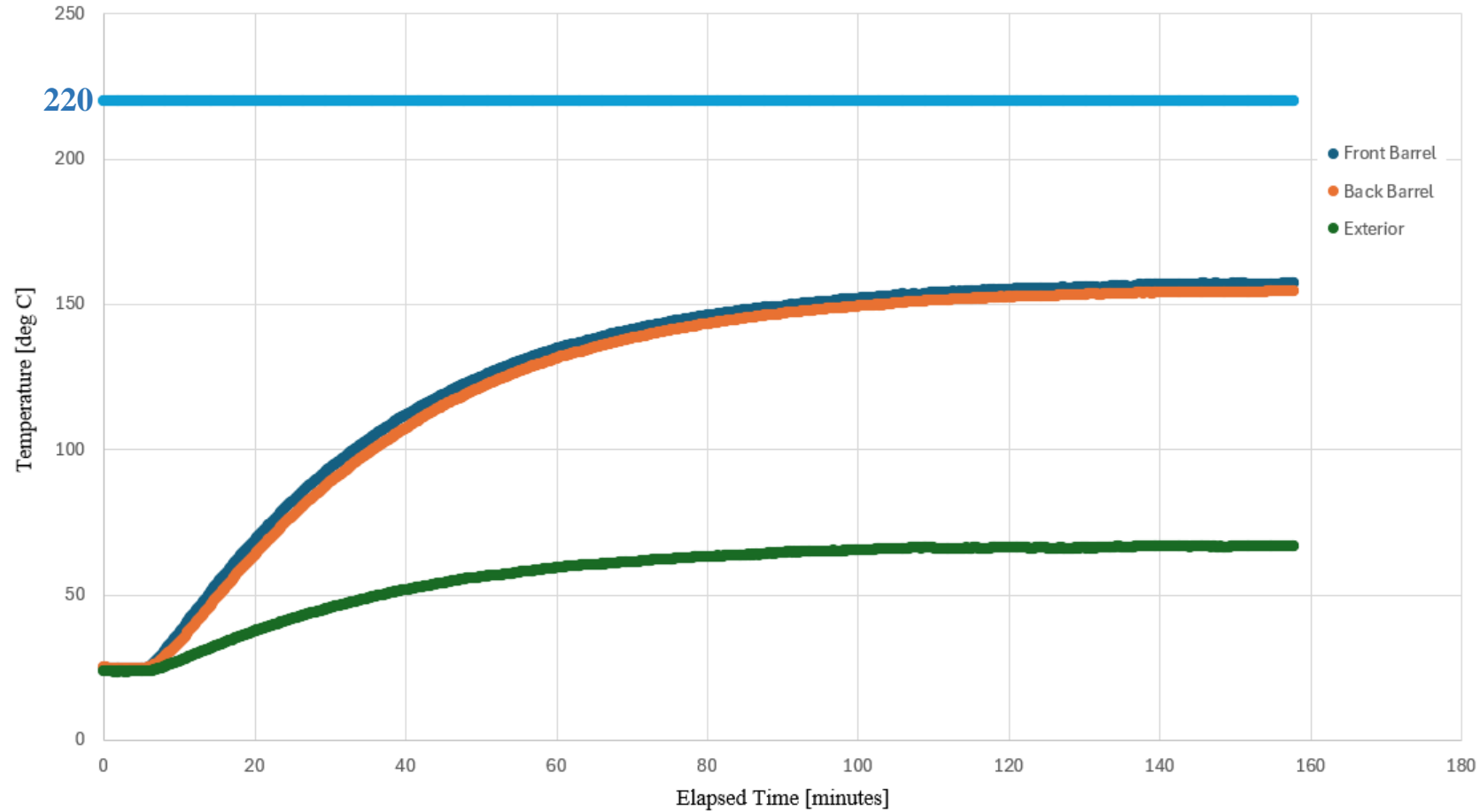
# Performance Evaluation/Testing Results

# Extruder Barrel Heating

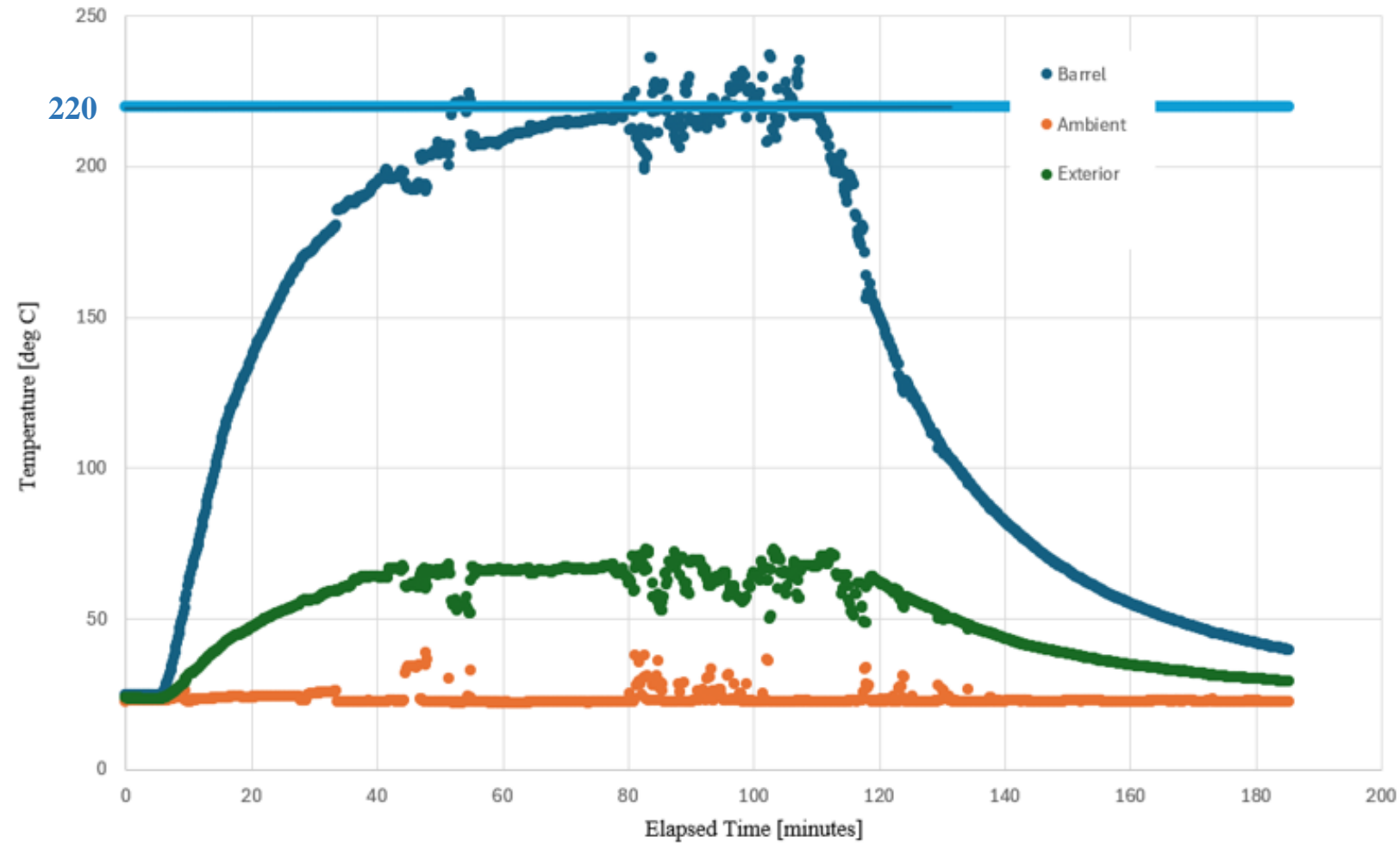
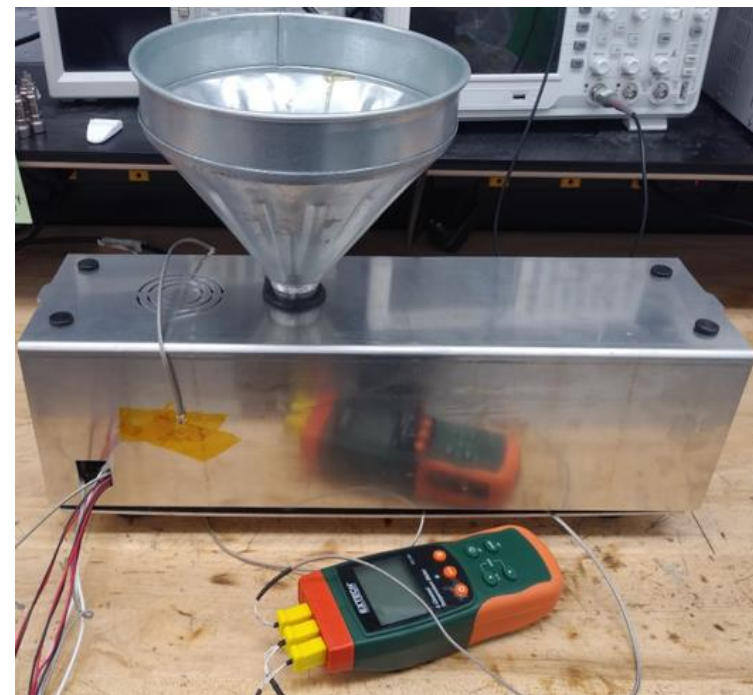




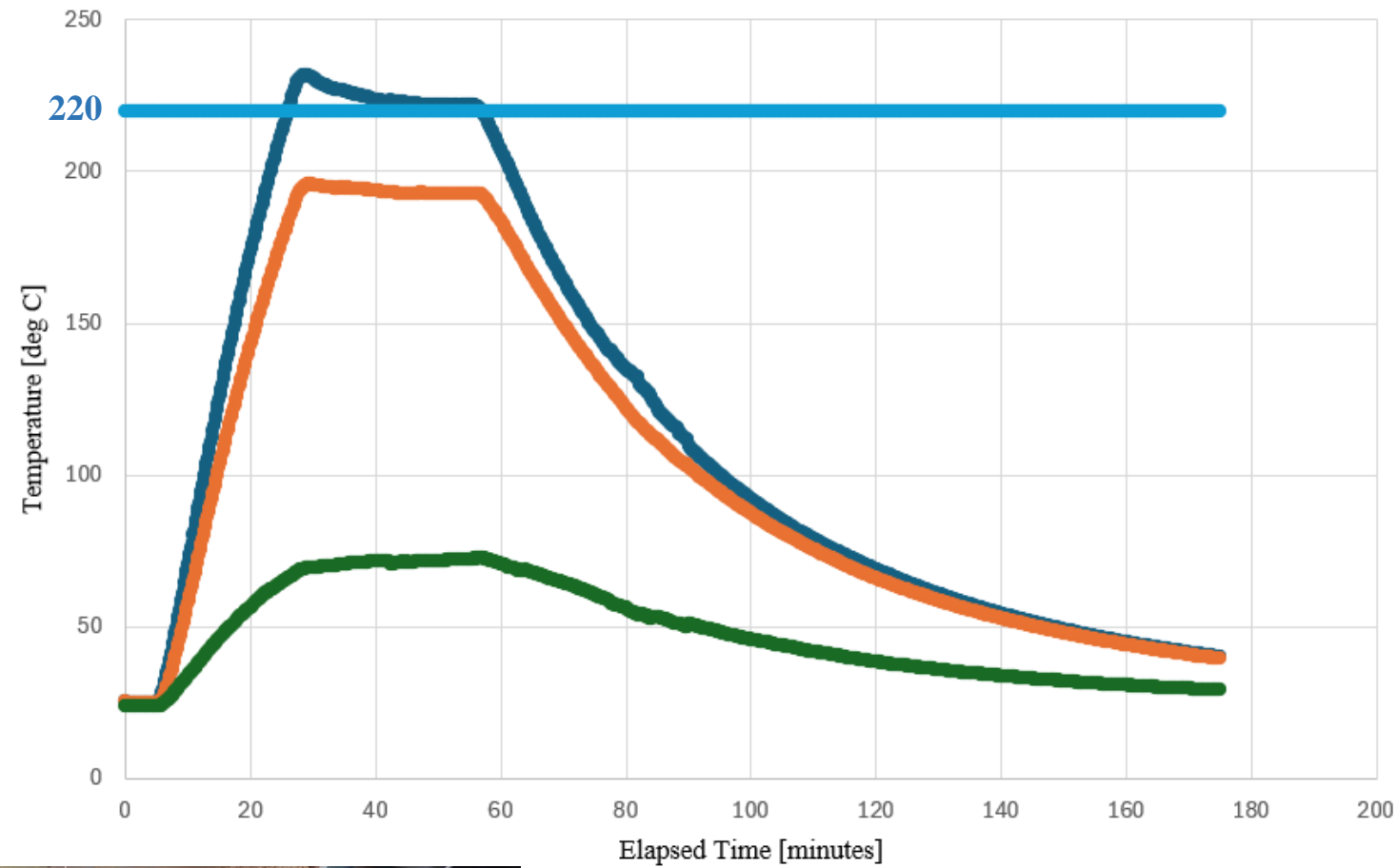
# Extruder Barrel Heating



# Extruder Barrel Heating



# Extruder Barrel Heating



# Melting Pellets

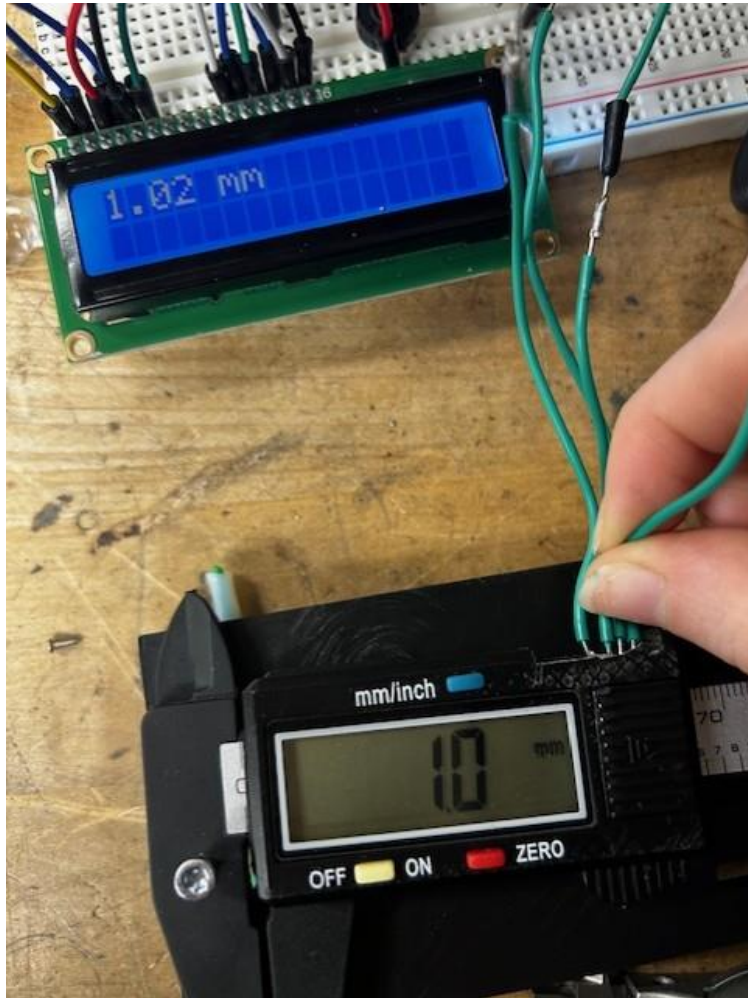


# Diameter Sensor: Wheels



- High grip, low force
- Rivets for filament

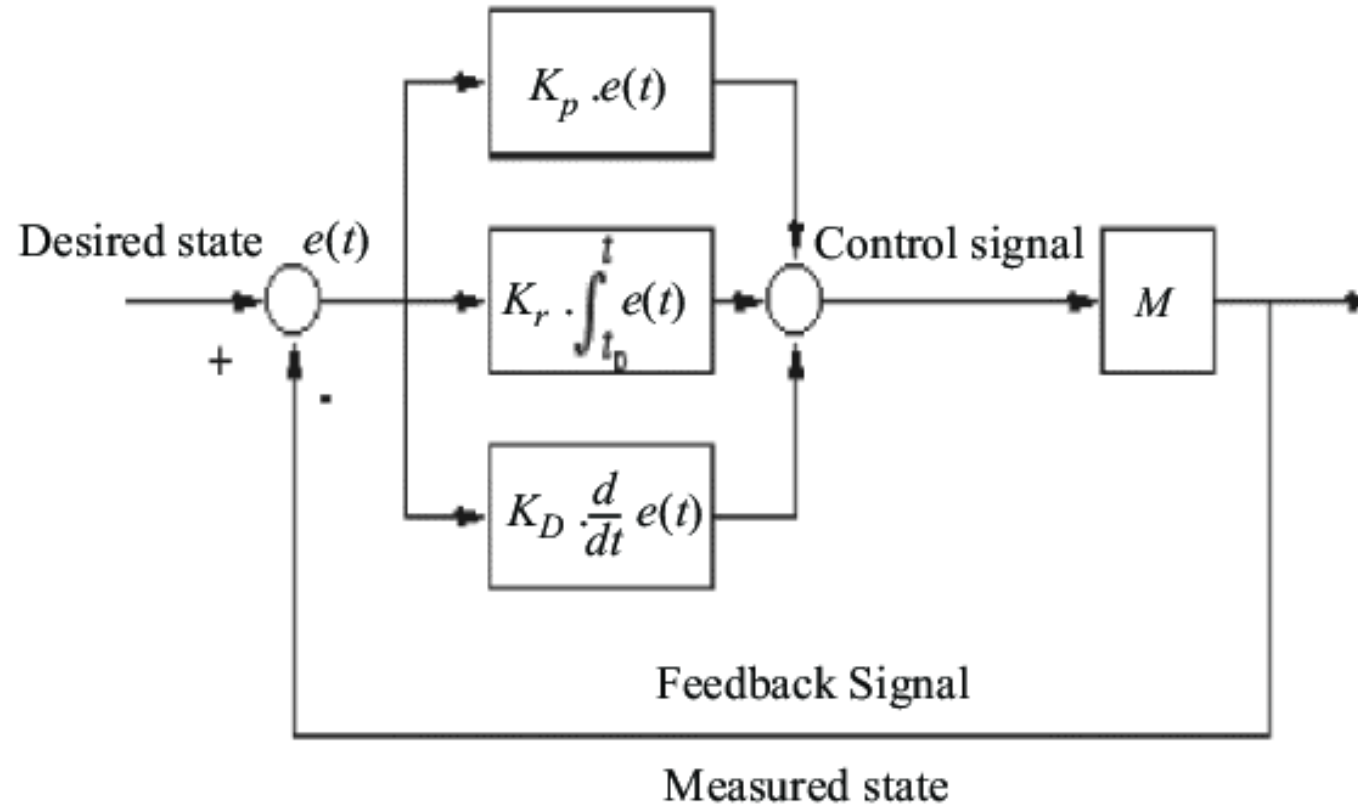
# Diameter Sensor: 2nd Prototype



- Automated Caliper
- 2 decimal places
- Bearing displacement
- Precise, but needs calibration
- Accessible and cheap
- Makes use of pre-existing technology

# Calculations

# PID Controller





# Thermodynamics/Heat Transfer – Barrel Heating

1. Customer Needs Statement 18 – Warm Up Rate
2. Needed to calculate energy in vs energy out, steady state
3. Generate 1<sup>st</sup> law equation:

$$\frac{dE}{dt} = W_{in,mechanical} + W_{in,electrical} - Q_{out} - \frac{dm}{dt}(H_{in} - H_{out})$$

4. Simplifies down to:

$$0 = (\tau\omega) + (VI) - (hA(T - T_{\infty})) - (\dot{m}C(T_{in} - T_{out}))$$

5. Solve for h (heat transfer coefficient) using 3 equations:

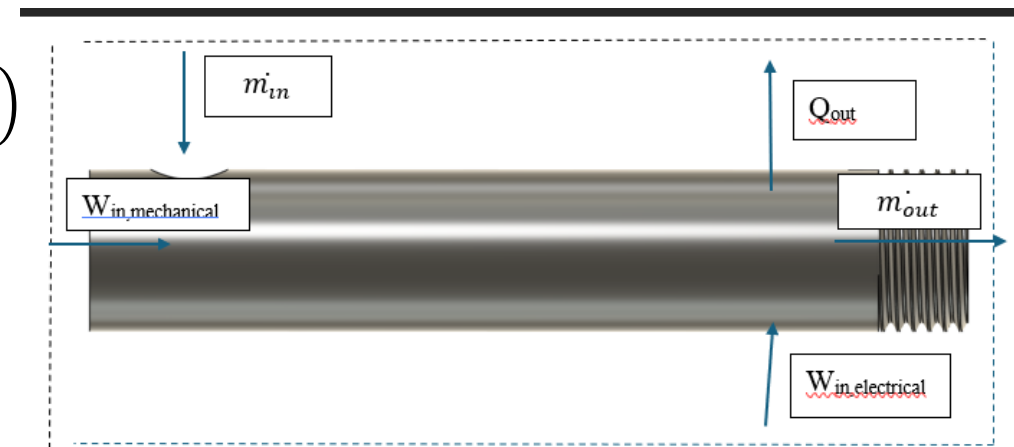
$$1. Nu = \frac{hD}{k}$$

$$2. Nu = 0.52 \left( Ra \frac{1}{4} \right)$$

$$3. Ra = \frac{g\beta\Delta TL^3}{\alpha\mu}$$

$$6. Ra = 24,040, Nu = 6.4751, h = 4.6615$$

$$7. I = 2.386 \text{ Amps}$$



## Thermodynamics/Heat Transfer – Barrel Heating

8. Barrel requires about 70kJ of energy (Q) to heat:

$$P = \frac{Q}{t}$$

9.  $P = W_{in,mechanical} + W_{in,electrical} = 39.1040 \text{ Watts}$

10. Solve the equation for t, time, and you get about 31.4 minutes required to heat up the barrel to 220°C steady state temperature

8. Barrel requires about 70kJ of energy (Q) to heat:

$$P = \frac{Q}{t}$$

9.  $P = W_{in,mechanical} + W_{in,electrical} = 39.1040 \text{ Watts}$

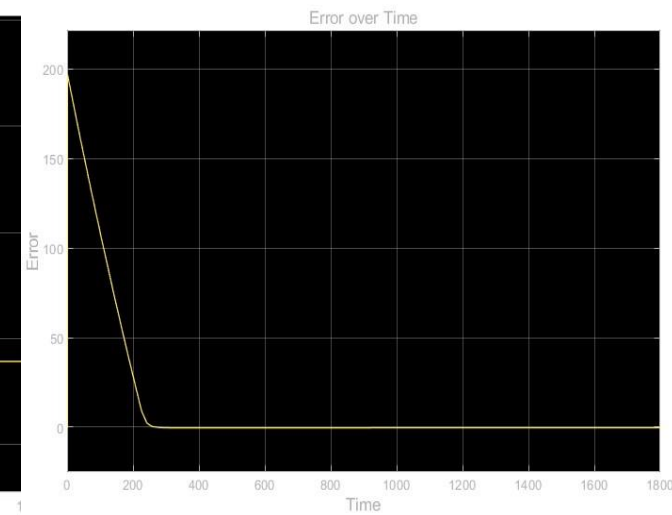
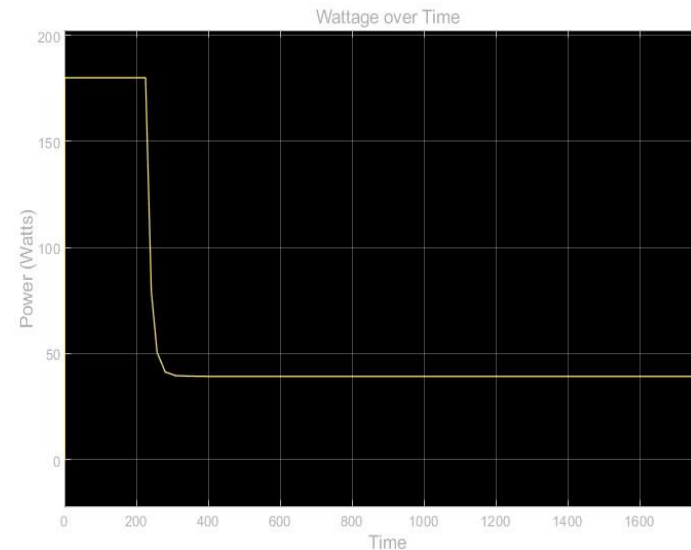
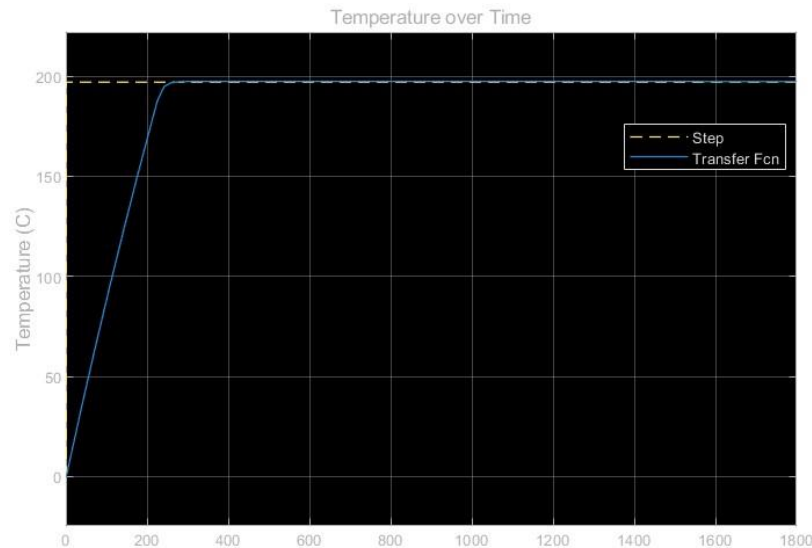
10. Solve the equation for t, time, and you get about 31.4 minutes required to heat up the barrel to 220°C steady state temperature

# PID Controller – Barrel Heating

- We can model the dynamics of the system through a transfer function in the form of:

$$G(s) = \frac{K_{DC}}{\tau s + 1}$$

- $\tau$  is the time it takes to reach 63% of the final steady state output = 968.3 seconds
- $K_{DC}$  is the relationship between the steady state output and constant input = 5.0378
- Using Simulink, we get the following gains:
  - $K_p = 15$
  - $K_I = 0.002$
  - $K_D = 0$

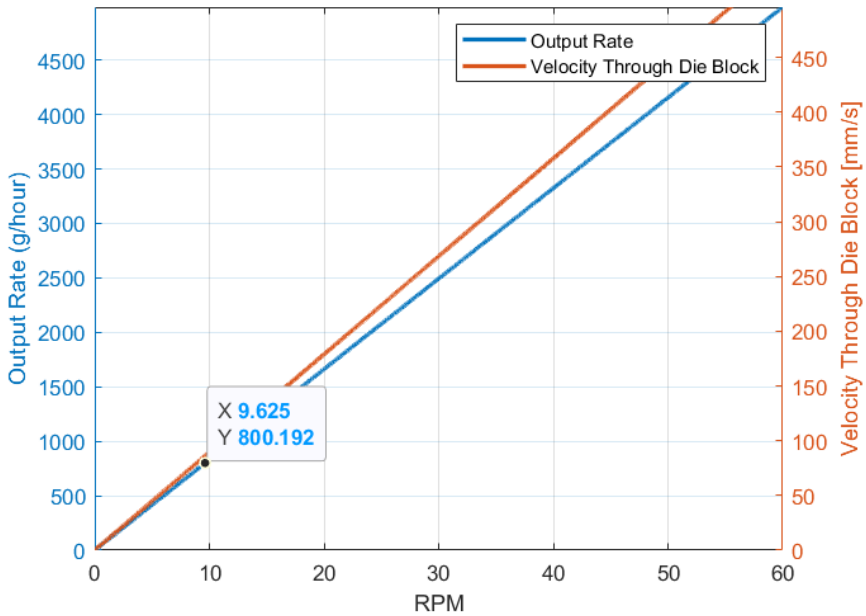


# Plastic Extrusion Analysis

Customer Needs Statement 1 – Able to maintain constant extrusion rate of 0.8 kg/hour

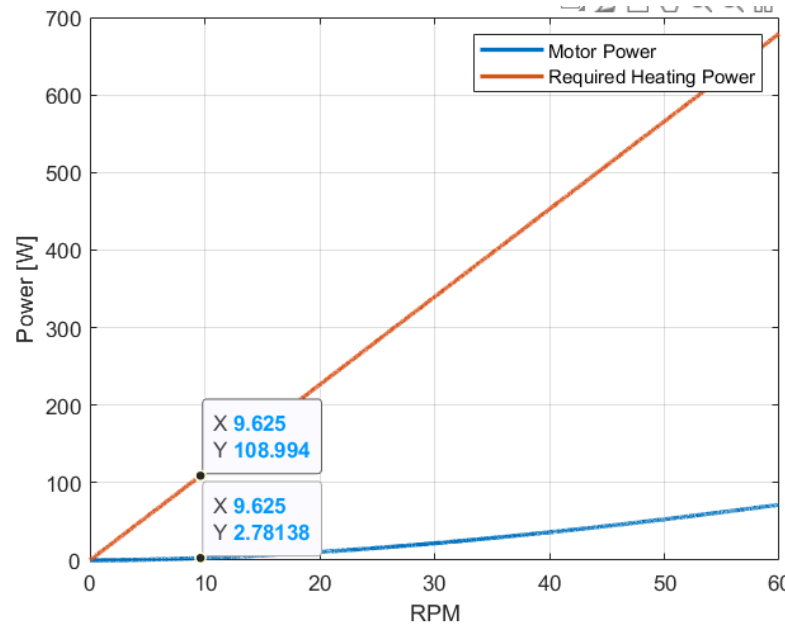


**Drag Flow Approximation**

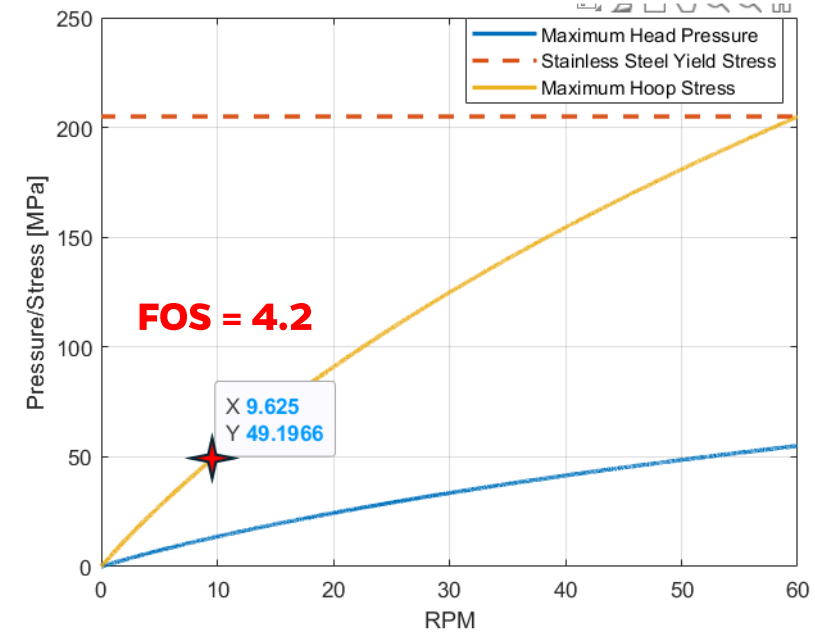


$$Q_d = 0.5\pi^2 * D^2 * N * h * \sin A * \cos A$$

**Required vs. Supplied Heating**

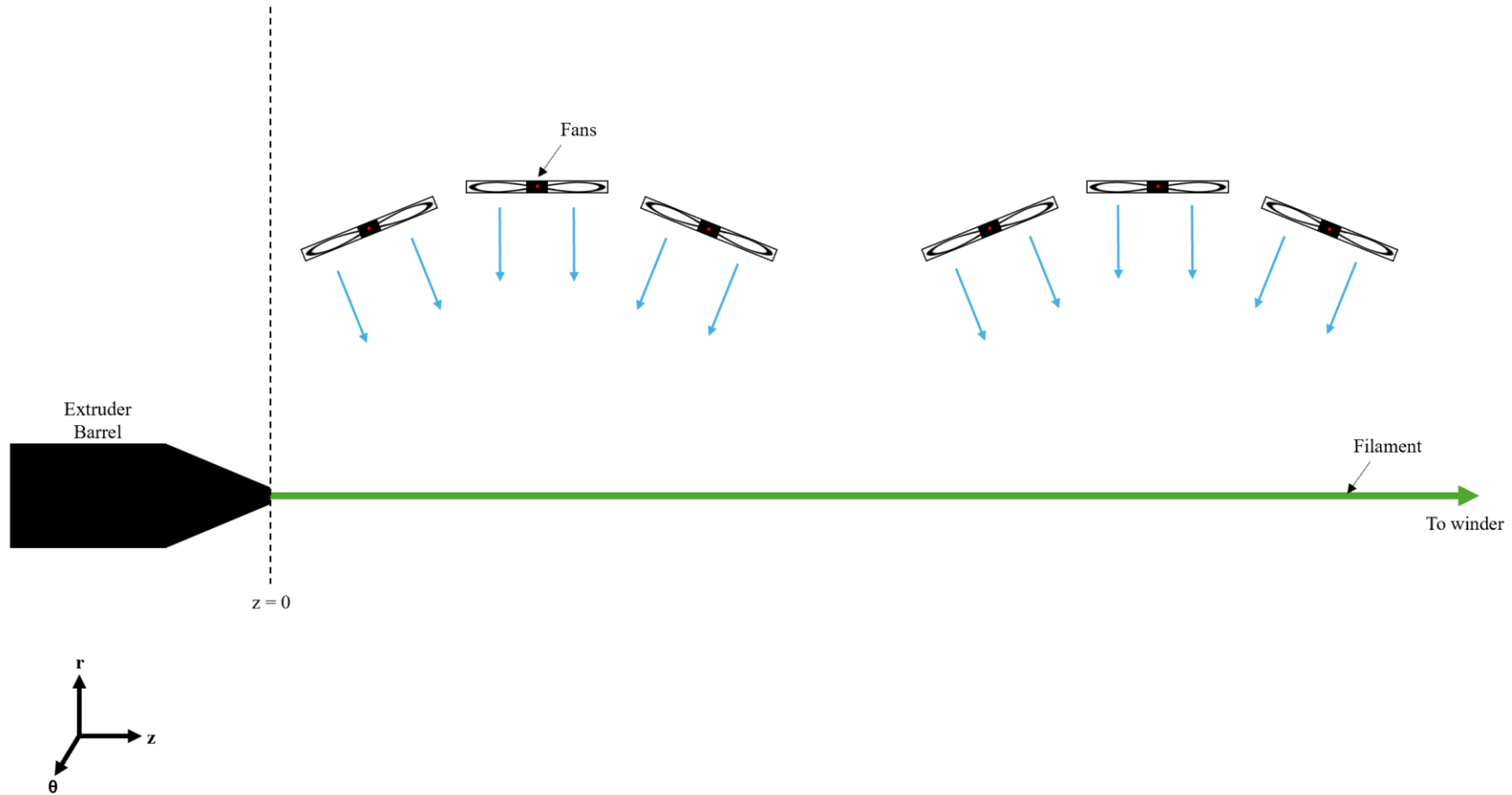


**Maximum Thick-Wall Hoop Stress**



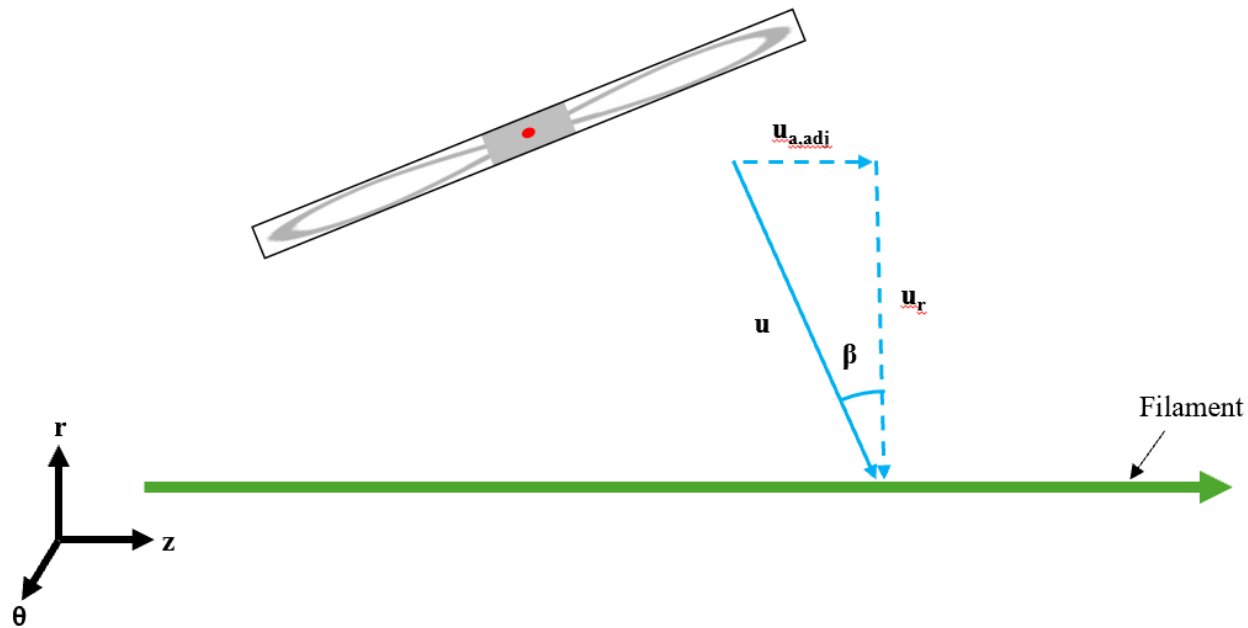
$$\sigma_{circumferential, maximum} = P_{max} \left( \frac{r_o^2 + r_i^2}{r_o^2 - r_i^2} \right)$$

# Cooling System Configuration



## Fan Flow Addition

- Found resultant velocity and direction (angle) interacting with the filament from all fans



$$u_a = \left(\frac{1}{n}\right) \sum_{i=1}^n u_{a,fan_i}$$

$$u_r = \left(\frac{1}{n}\right) \sum_{i=1}^n u_{r,fan_i}$$

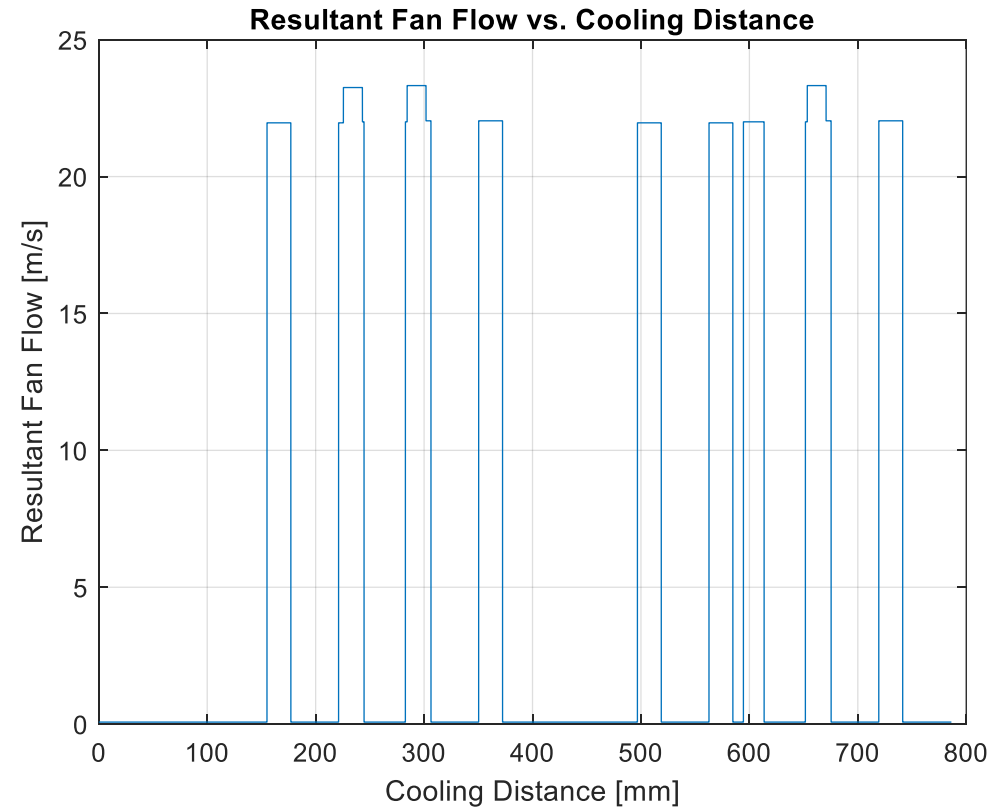
$$u_{a,adj} = u_a - u_{ext}$$

$$u = \sqrt{u_{a,adj}^2 + u_r^2}$$

$$\beta = \tan^{-1} \left( \frac{u_{a,adj}}{u_r} \right)$$

# Fan Flow Addition

- Resultant flow velocity acting on filament over cooling path



# Determination of Flow Properties

- Flow parameters used for heat transfer analysis found using empirical Nusselt number equation

$$Re = \frac{uD_{fil}}{\nu}$$

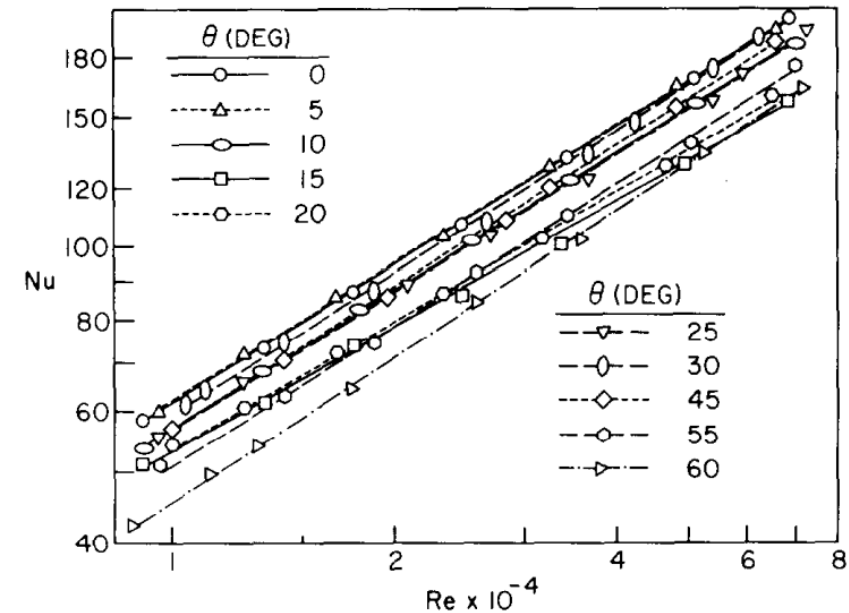
$$Nu = CRe^n$$

$$h = \frac{Nu * k}{D_{fil}}$$

Table 2. The constants  $C$  and  $n$  for equation (5)

$\theta$ (deg.)	$C$	$n$
0	0.207	0.618
5	0.217	0.614
10	0.200	0.614
15	0.302	0.561
20	0.270	0.574
25	0.187	0.620
30	0.172	0.634
45	0.161	0.636
55	0.152	0.630
60	0.113	0.651

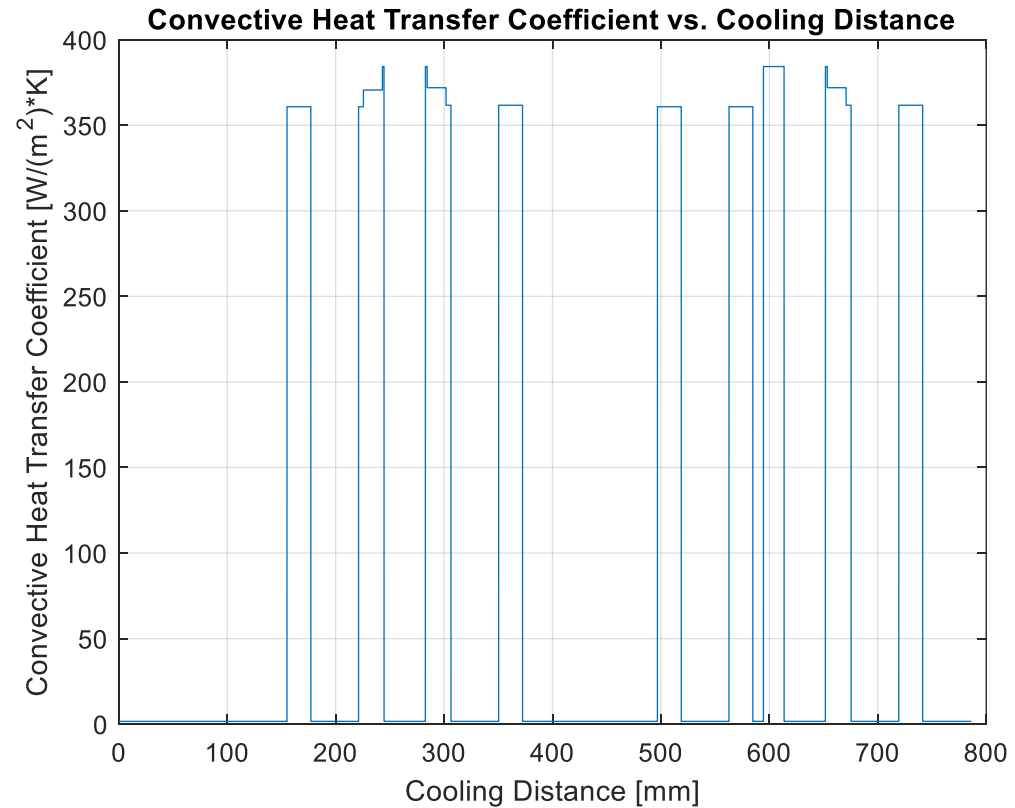
Effect of yaw on forced convection heat transfer





# Determination of Flow Properties

- Convective heat transfer coefficient over length of cooling path



## Lumped Capacitance

- Lumped capacitance is invalid because  $Bi > 0.1$  for predicted fan flow of 22 m/s

$$Bi = \frac{h_{max} r_o}{k_{PLA}}$$

$$Bi = \frac{(384.31 \text{ W/m}^2 \cdot \text{K})(8.75 \times 10^{-4} \text{ m})}{(0.183 \text{ W/m} \cdot \text{K})} = 1.838$$

## Exact Solution

- Exact solution is invalid because external flow conditions change with time
- Iterative solution requires updating non-uniform initial conditions, however, the exact solution assumes uniform initial conditions

$$\frac{T - T_{\infty}}{T_i - T_{\infty}} = \sum_{n=1}^{\infty} C_n \exp(-\zeta_n^2 Fo) * J_0(\zeta_n^2 r^*)$$

$$C_n = \frac{2}{\zeta_n} \left( \frac{J_1(\zeta_n)}{J_0^2(\zeta_n) + J_1^2(\zeta_n)} \right)$$

$$\zeta_n \left( \frac{J_1(\zeta_n)}{J_0(\zeta_n)} \right) = Bi$$

$$Bi = \frac{hr_o}{k}$$

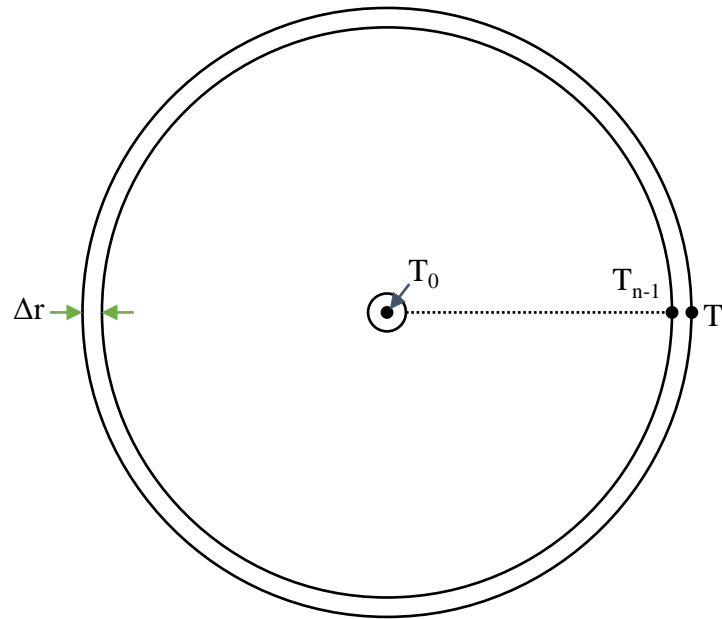
$$Fo = \frac{\alpha t}{r_o^2}$$

$$\alpha = \frac{k}{\rho c_p}$$

$$r^* = \frac{r}{r_o}$$

## Finite Difference Method - Discretization

- Heat transfer problem was discretized with respect to both time and filament radius
- Temperature is assumed to be constant at each filament radius
- Easy transformation from time elapsed to position along cooling length:  $\Delta z = u_{ext}(\Delta t)$



## Finite Difference Method - Explicit

- Finite difference methods were used to find temperature distribution across the filament over time
- Explicit method predicts temperature at the next time step directly (explicitly) using the temperature distribution at the previous time step
- Conceptually simpler but requires satisfying convergence conditions
- The maximum time step for convergence is the largest time step that satisfies all convergence conditions

## Finite Difference Method - Explicit

- The following are the explicit finite difference method equations and their convergence conditions:

Outer Surface: 
$$T_n^{p+1} = 2Fo \left[ \left( \frac{r_n Bi}{r_{n-1}} \right) T_\infty + T_{n-1}^p \right] + \left( 1 - 2Fo - \frac{2r_n Bi Fo}{r_{n-1}} \right) T_n^p$$

$$\Delta t \leq \frac{(\Delta r)^2}{2\alpha \left( 1 + \frac{r_n Bi}{r_{n-1}} \right)}$$

Other Elements: 
$$T_i^{p+1} = \left( \frac{Fo(\Delta r + 2r_i)}{2r_i} \right) T_{i+1}^p + (1 - 2Fo) T_i^p + \left( \frac{Fo(2r_i - \Delta r)}{2r_i} \right) T_{i-1}^p$$

$$\Delta t \leq \frac{(\Delta r)^2}{2\alpha}$$

Central Element: 
$$T_0^{p+1} = Fo \left( T_2^p - 2T_1^p \right) + (Fo + 1) T_0^p$$

$$\Delta t \geq \frac{-(\Delta r)^2}{\alpha} \quad (\text{always satisfied!})$$

## Finite Difference Method - Implicit

- There are no convergence conditions for the implicit method, meaning less computation time without sacrificing much or any accuracy
- The following are the implicit finite difference method equations:

Outer Surface: 
$$\left(1 + 2Fo + \frac{2r_n BiFo}{r_{n-1}}\right) T_n^{p+1} - 2Fo T_{n-1}^{p+1} = T_n^p + \left(\frac{2r_n BiFo}{r_{n-1}}\right) T_\infty$$

Other Elements: 
$$\left(\frac{-Fo(\Delta r + 2r_i)}{2r_i}\right) T_{i+1}^{p+1} + (1 + 2Fo) T_i^{p+1} + \left(\frac{Fo(\Delta r - 2r_i)}{2r_i}\right) T_{i-1}^{p+1} = T_i^p$$

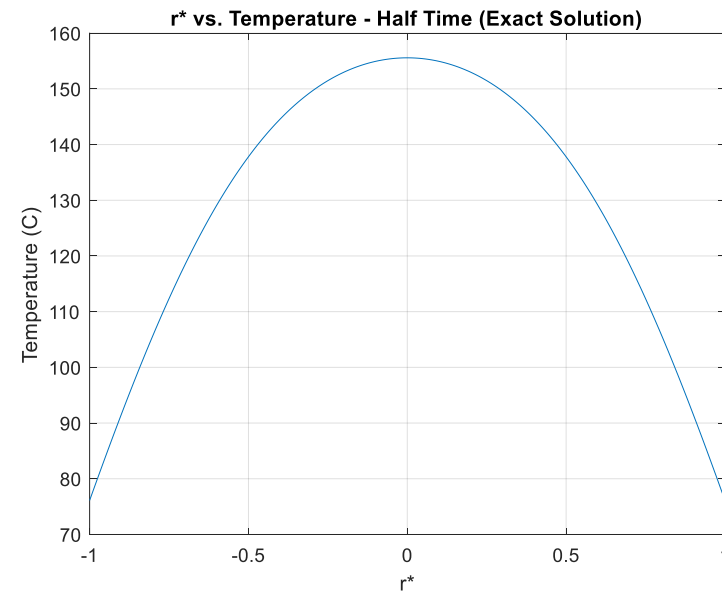
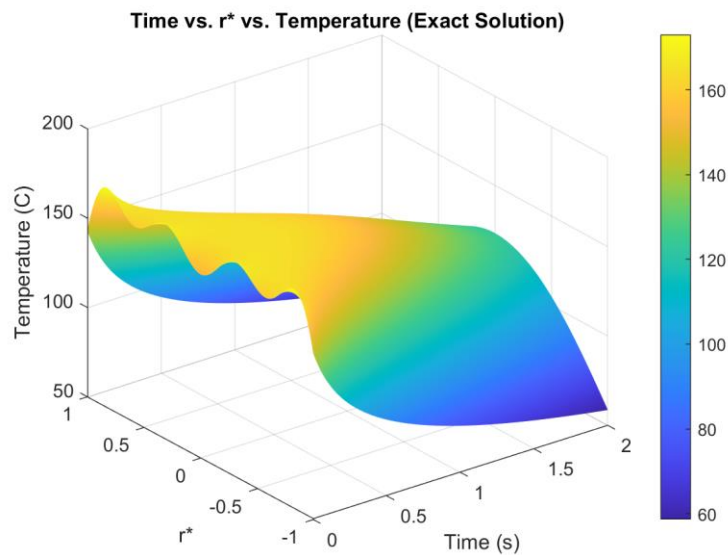
Central Element: 
$$(2Fo + 1) T_0^{p+1} - 2Fo T_1^{p+1} = T_0^p$$

coefficients                      constants (including  $T^p$  values)

$\{T^{p+1}\} = [A]^{-1} \{C\}$

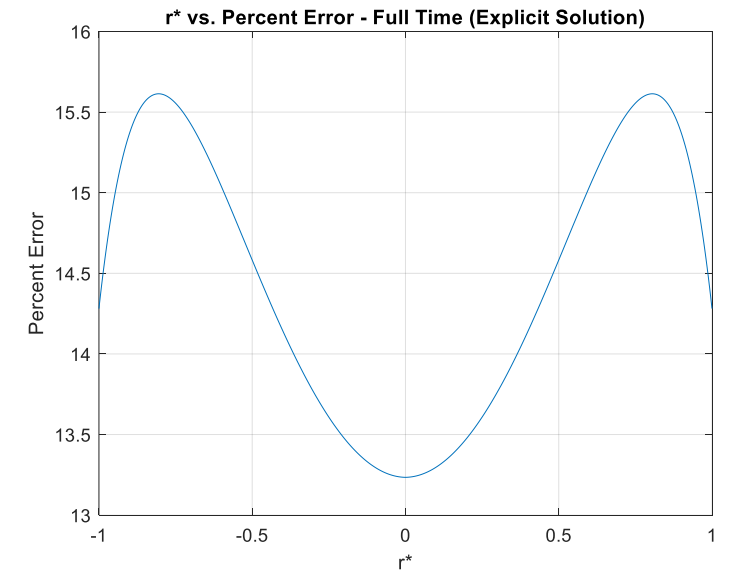
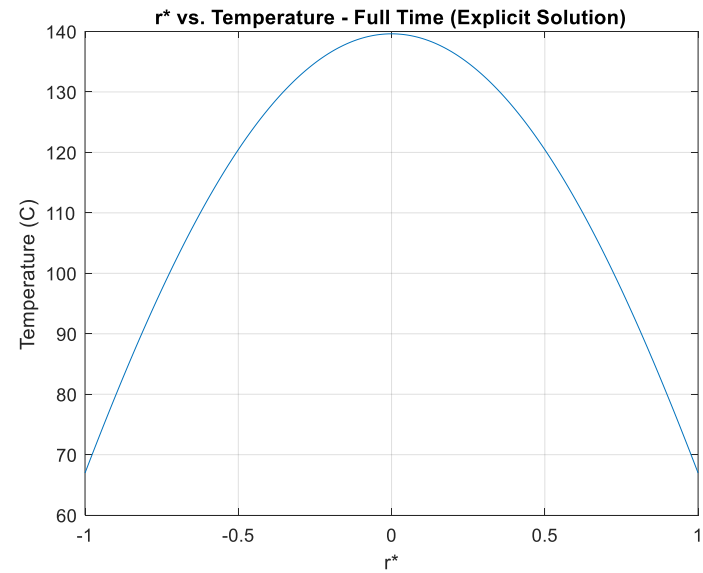
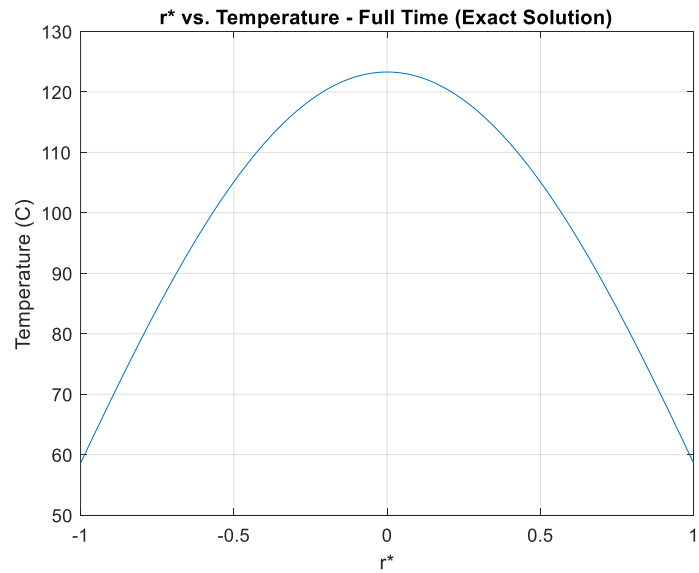
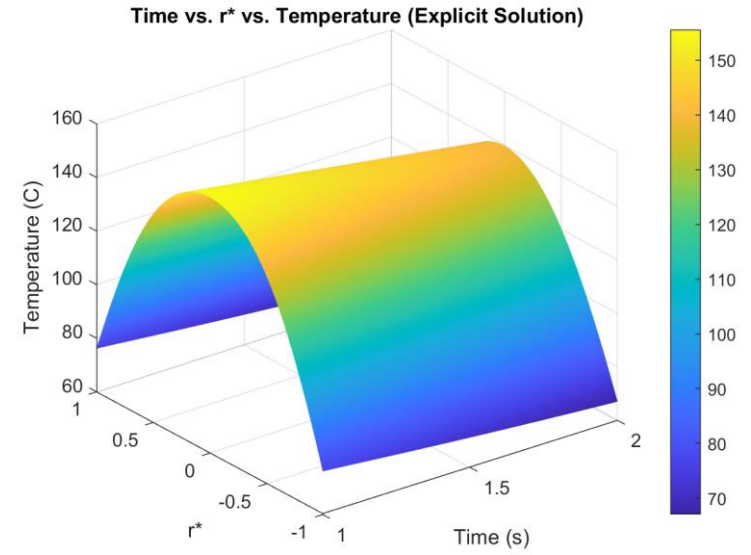
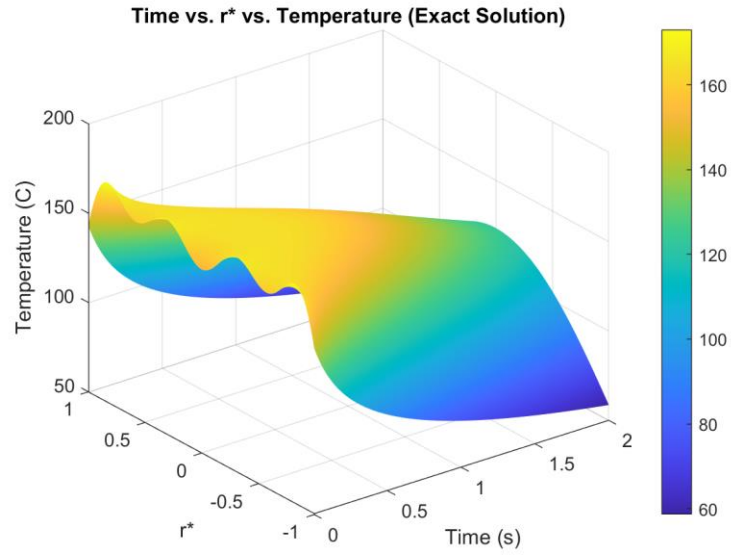
## Test Scenario

- A test scenario with constant external conditions was performed to assess the validity of the explicit and implicit finite difference methods
- Similar conditions to the actual cooling scenario used for test
- The total test time was 2 seconds, and the exact temperature profile at  $t = 1$  sec was used as the initial condition for the finite difference heat transfer methods

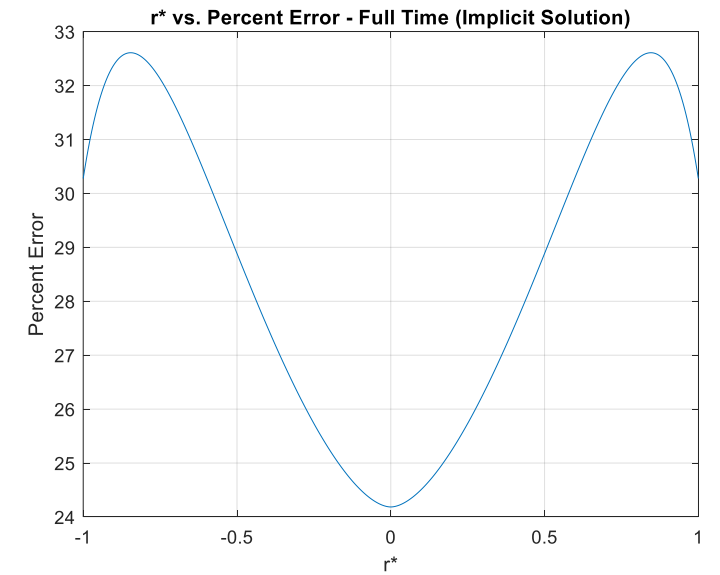
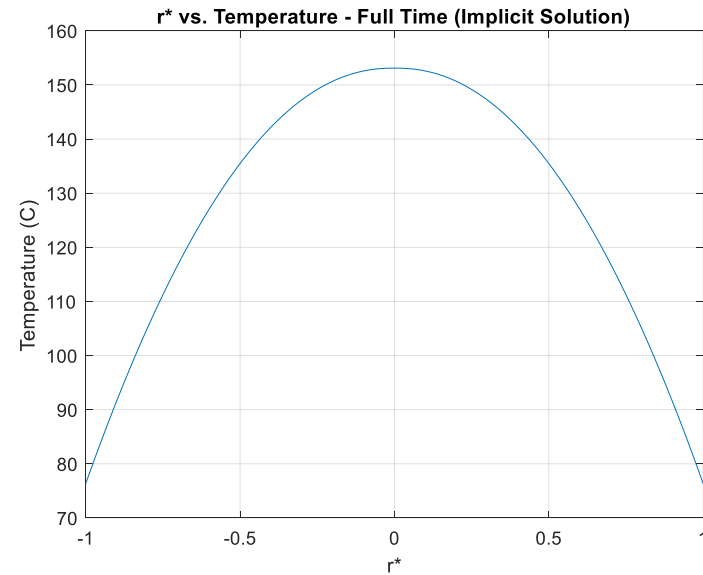
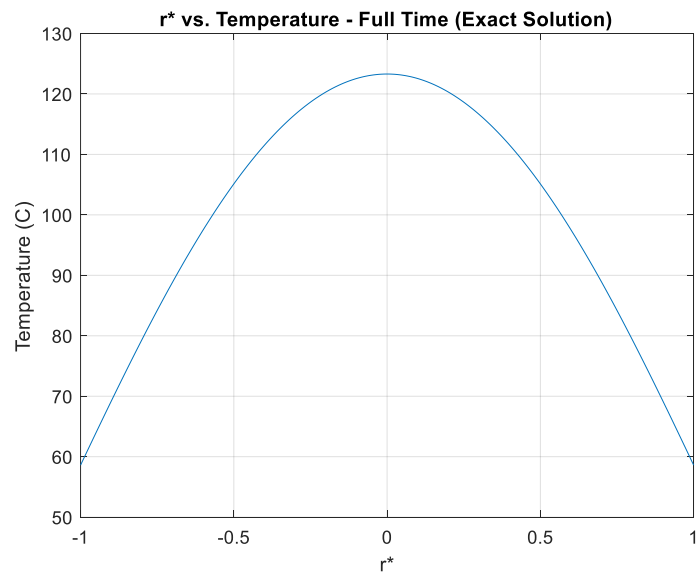
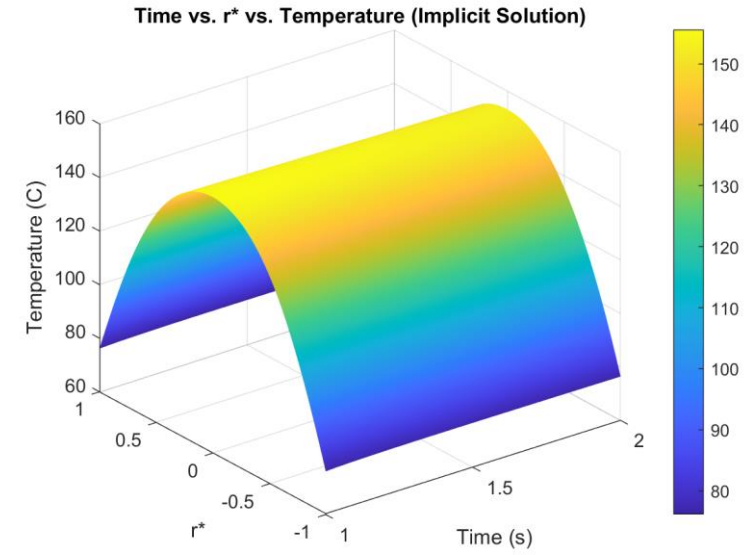
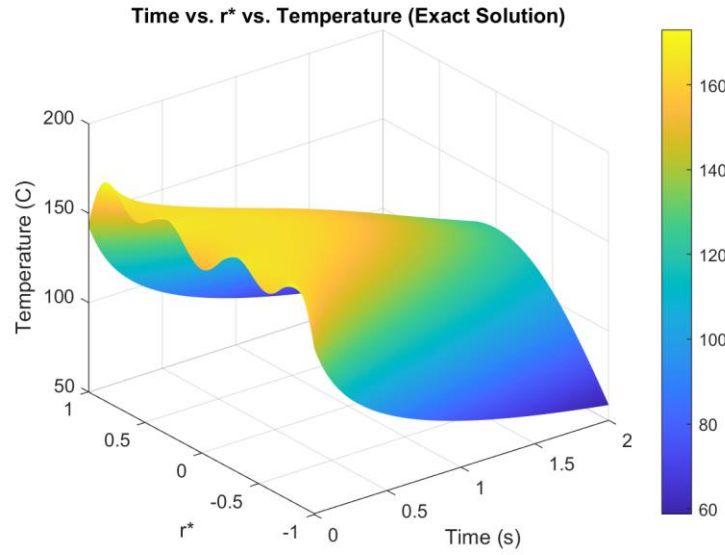




# Test Scenario - Explicit Method

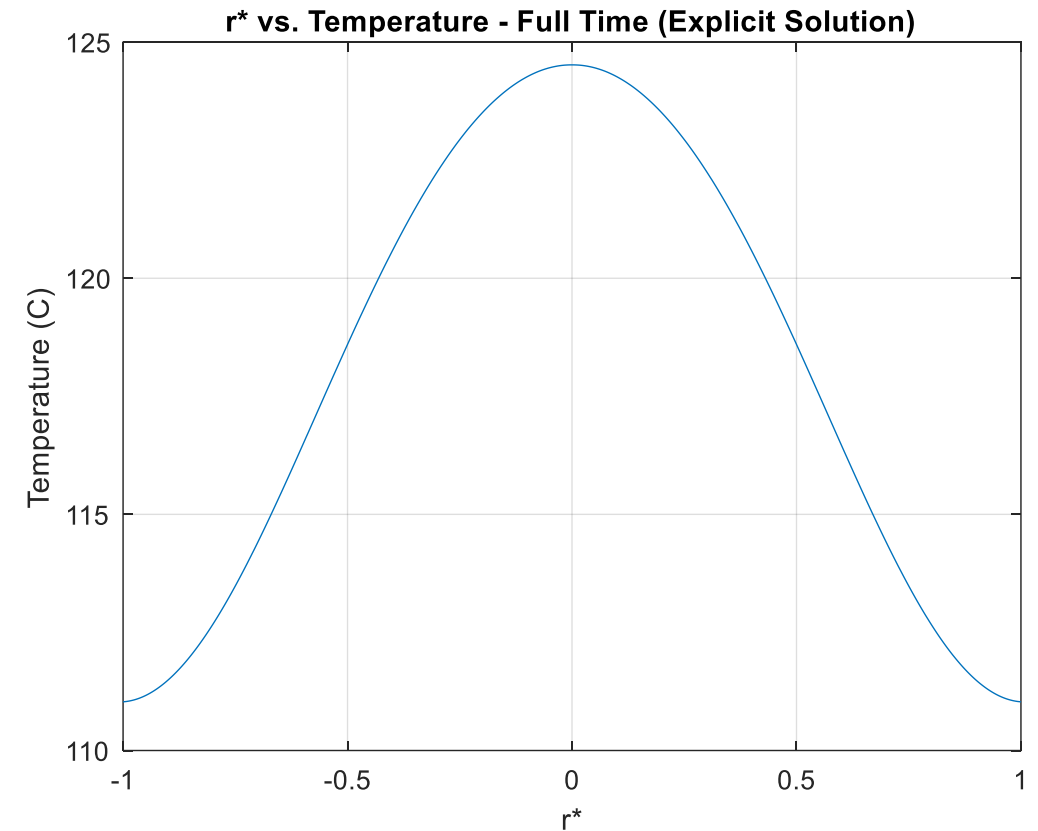
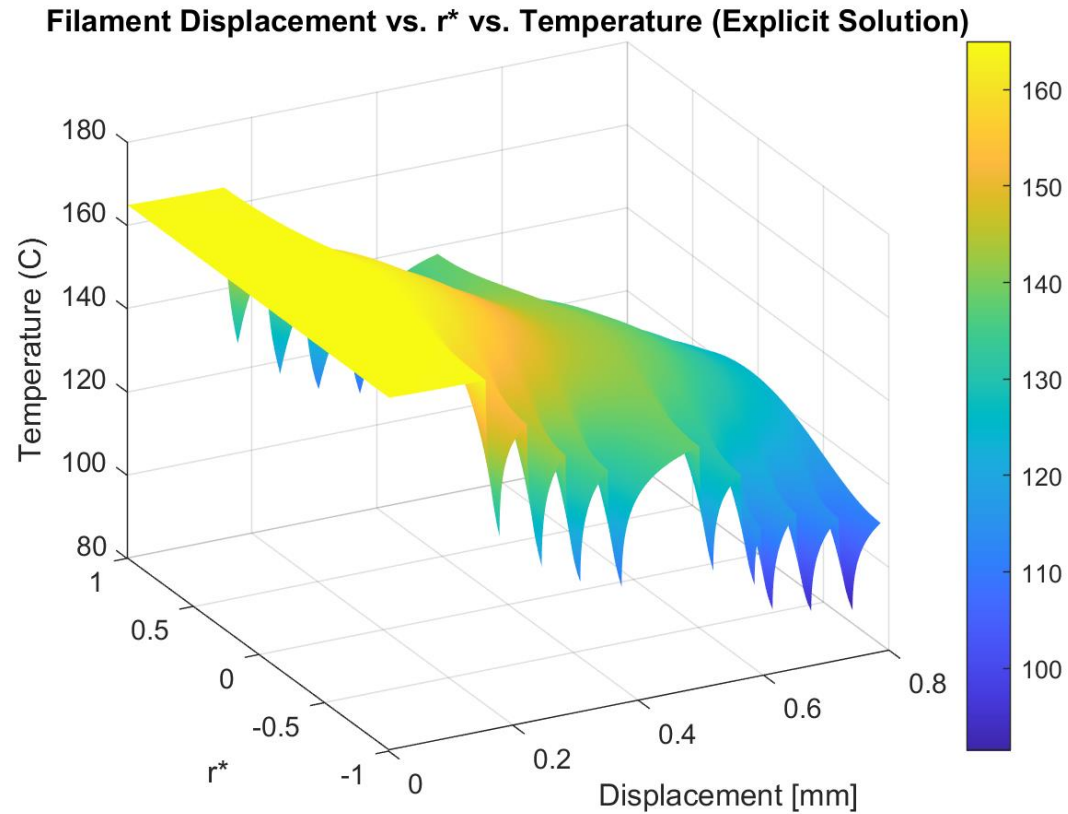


# Test Scenario - Implicit Method



# Filament Cooling - Explicit Method

- Temperature distribution along cooling path calculated using explicit transient heat transfer:



# Improvements



- Time, Testing, and Tender!
- Trying new sensors
- Create winder based on extruder
- Adding controls for other filament types
- Certain parts manufactured over 3D printed
- Redo extruder design, but reverse engineer Artme3D first

# What Makes Our Design Unique

- Diameter sensor is recyclable!
  - Build your own replacement and customizable
- Spool
- 3-D printed parts: Affordability

**UF** | Herbert Wertheim  
College of Engineering  
UNIVERSITY *of* FLORIDA

---

POWERING THE NEW ENGINEER TO TRANSFORM THE FUTURE

# Winder Calculations (For Controls)

- Follows Customer Need of 800 g/hr at a diameter of 1.75 mm

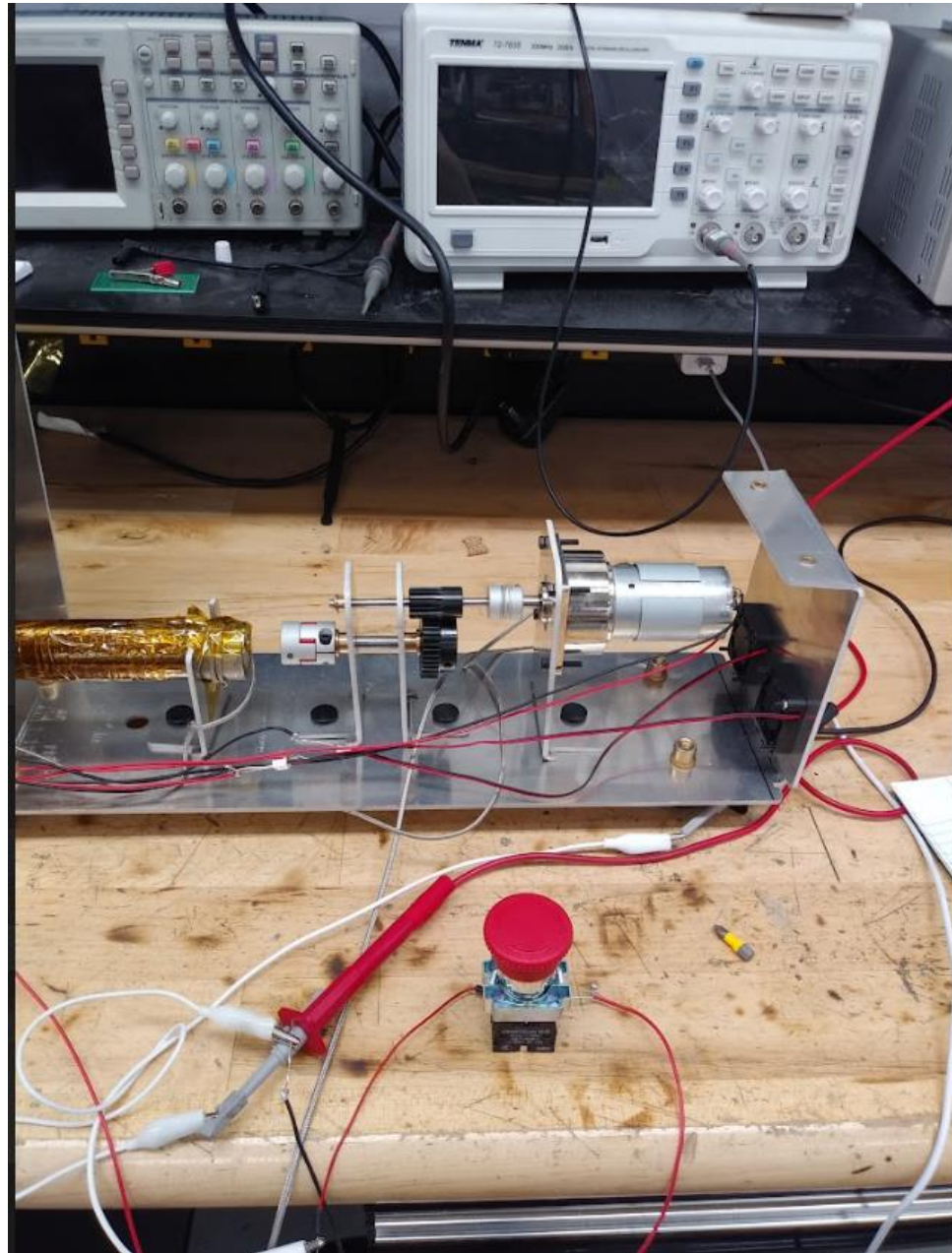
$$\frac{\text{Extrusion Rate} \left(\frac{g}{hr}\right)}{\text{Density} * \text{Area}} = \frac{\frac{800 g}{1 hr} * \frac{1 hr}{60 min}}{\frac{0.001252 g}{1 mm^3} * \pi \left(\frac{1.75 mm}{2}\right)^2} = 73.911 \frac{mm}{min}$$

- Assumes no slipping of gears/belt
- Assumes constant filament density and radius
- Density=1.252 g/cm<sup>3</sup>
- Spool Circumference =49.5046 mm

$$\frac{\text{Extrusion Rate} \left(\frac{mm}{min}\right)}{\text{Circumference} (mm)} = \frac{73.911 \frac{mm}{min}}{2\pi * 49.5046 mm} = 12.471 rpm$$

$$\frac{\partial}{\partial r} (RPM) = -\frac{73.911 \frac{mm}{min}}{2\pi * r^2} = -\frac{11.763}{r^2} = \text{Rate of decreasing rpm}$$

# E-Stop





**Calculate:**  
 Volts  Temp

**Resistance Wire**  
 Nichrome 60 ▾  
 Gauge: 28 Length (inches): 36

Transformer Volts Out: 25.7

**Copper Lead Wire**  
 Gauge: Length (feet):

**Resistance Wire**  
 Ω/Ft: 4.24 Ohms: 12.71 Volts: 25.7  
 Dia (in): 0.0126 Dia (mm): 0.321

**Copper Wire**  
 Ω/Ft: NaN Ohms: 0 Volts: 0

Amps: 2 Watts: 52  
 Temp °F: 830 °C: 446

**Calculate:**  
 Volts  Temp

**Resistance Wire**  
 Nichrome 60 ▾  
 Gauge: 28 Length (inches): 85

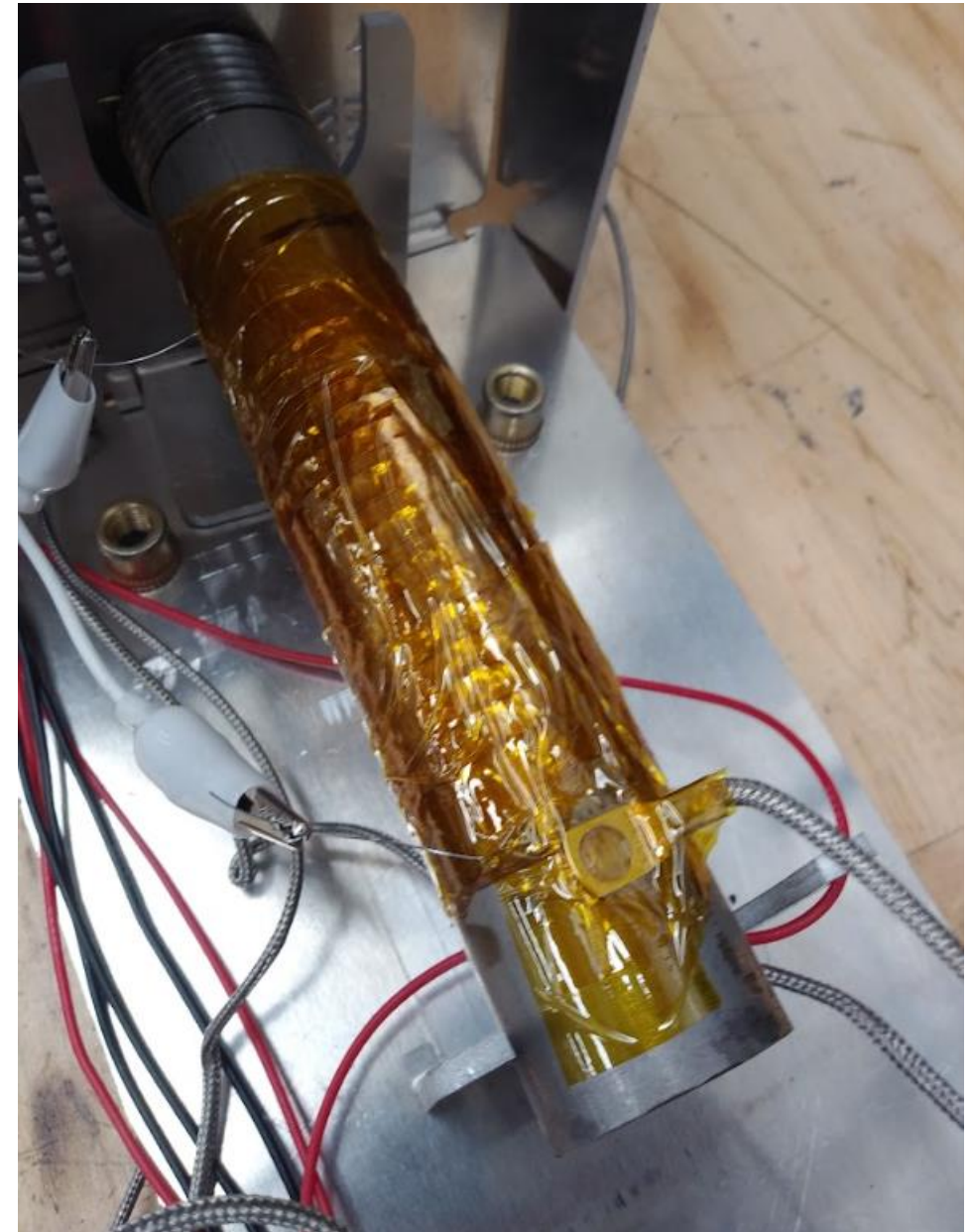
Transformer Volts Out: 60

**Copper Lead Wire**  
 Gauge: Length (feet):

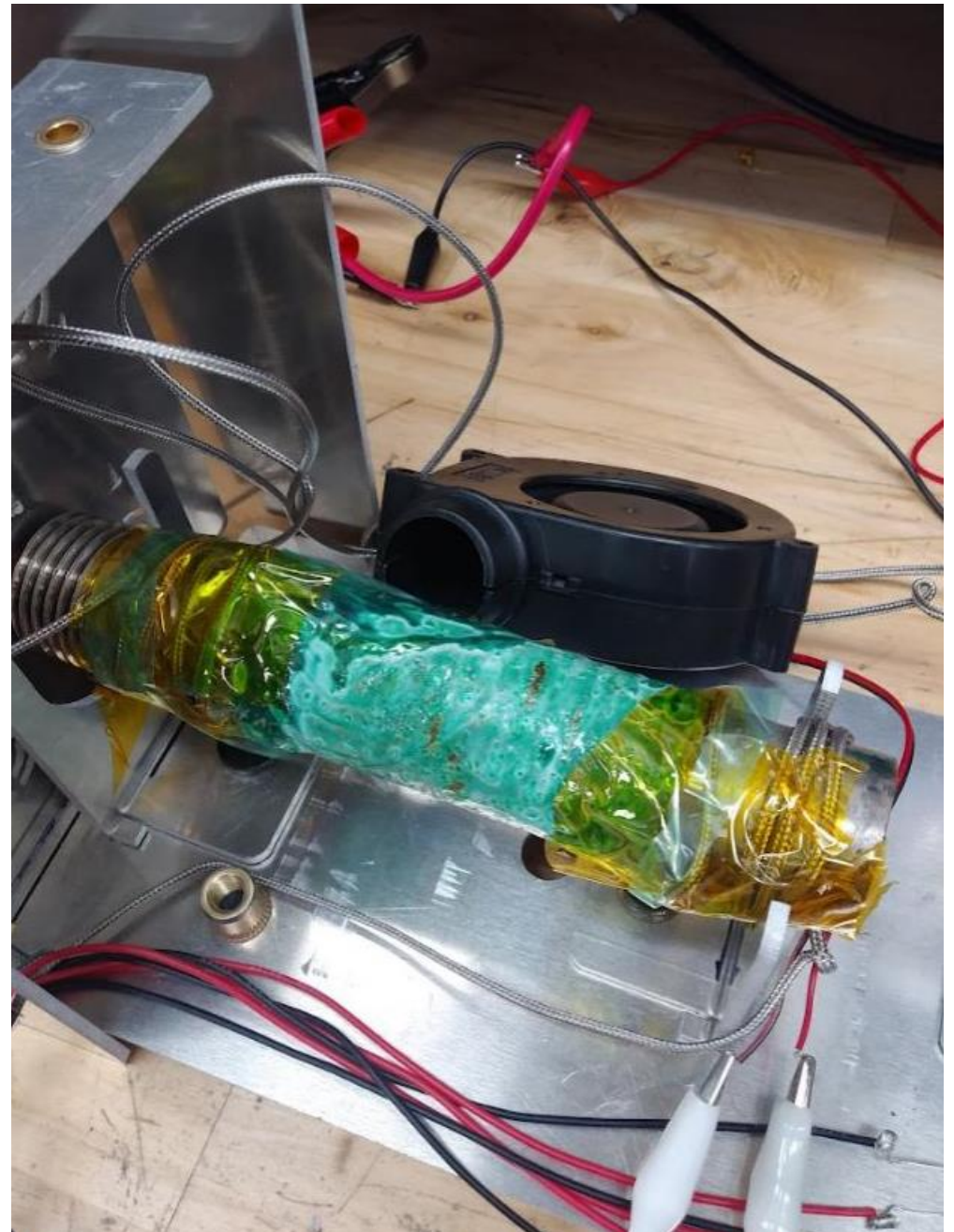
**Resistance Wire**  
 Ω/Ft: 4.24 Ohms: 30 Volts: 60  
 Dia (in): 0.0126 Dia (mm): 0.321

**Copper Wire**  
 Ω/Ft: NaN Ohms: 0 Volts: 0

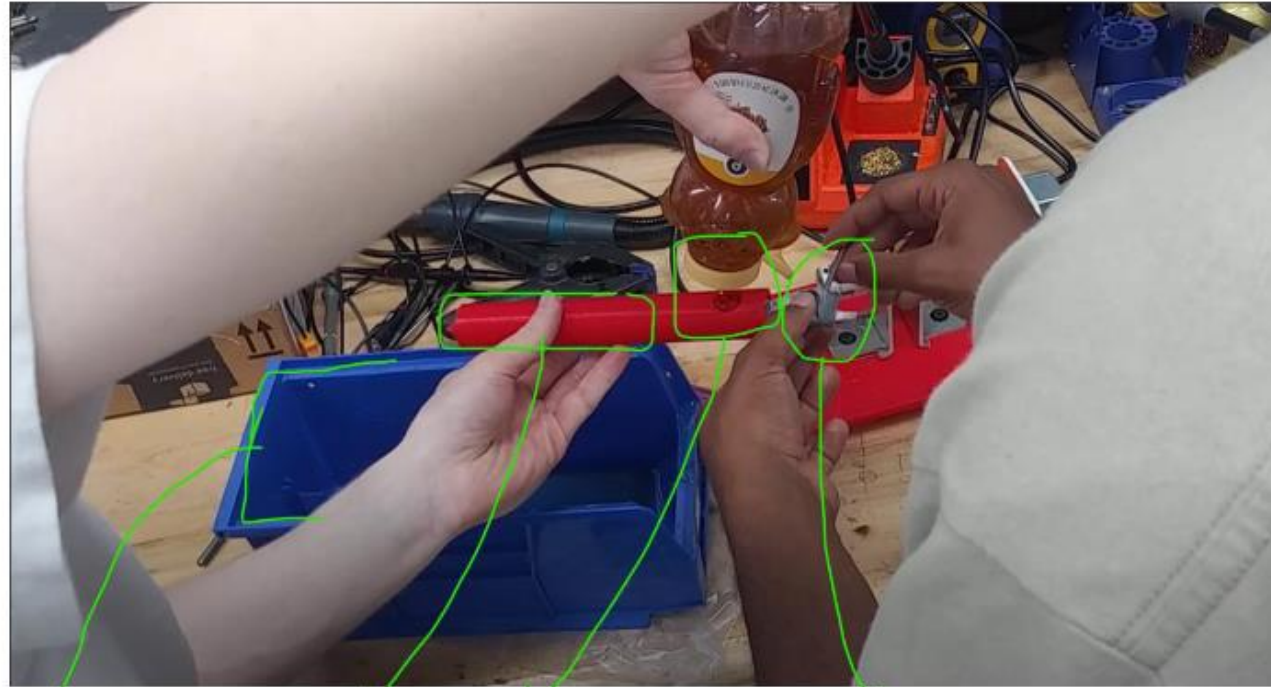
Amps: 2 Watts: 120  
 Temp °F: 820 °C: 439







# Honey



BUCKET 1:1 SCALE MODEL OVERFILLING MAKESHIFT HAND CRANK

$$Re = \frac{\rho v L}{\mu}$$

$$\dot{V} = 0.5 * \pi * D^2 * \omega * \sin(\alpha) * \cos(\alpha)$$

Trial	RPM	Elapsed Time	Actual Output	Expected Output	Percent Difference
1	30	254 s	128 g	0.366 g	199%
2	5	211 s	30 g	0.05 g	199%

# Quote 1637-453

[Upload](#)[Configure](#)[Review](#)[Quote Options](#) ▾[Checkout Now](#)

CNC Machining 1 Part

ITAR/Export Controlled? No [Change](#)

Tighter tolerances, more materials and finishes from Protolabs Network. [Get an instant quote](#) ✕



## REV B SCREW.STEP

1231-8834-001

Current Revision: 1

Stainless Steel 303

Edges broken (tool marks visible)

Mill

X: 194.79mm Y: 20.20mm Z: 20.20mm

Machining Tolerance: +/- 0.005 in. (0.13mm)

[View Analysis](#)[Configure Part](#)[Upload Revision](#)[Part Options](#) ▾

Quantity

− 1 +

1 Part @ \$3,345.87

\$3,345.87

**Total**

**\$3,345.87**

Order by:

Mon 8:10 PM

Receive by:

Tue, Aug 13

Standard

Tue, Aug 13

Economy

Thu, Aug 22

Economy

Fri, Aug 30

- \$817.69

- \$1,051.32

⚠ View analysis details & approve ⓘ

[+ New Quote](#)[☰ Quotes](#)[☰ Parts & Molds](#)[🕒 Order History](#)[📖 Design Guides](#)[📞 Contact Us](#)[+ Give Us Feedback](#)[👤 Profile & Settings](#)[➔ Sign Out](#)[Protolabs Companies](#)[Protolabs Network](#) ✕[myRapid](#) ✕[📢 Whats New](#)

# Quote Q90-1696-2433

+ Add Certifications

Download

Share

1 Part Uploaded

Collapse All

Recent Activity Log

1



### REV B SCREW.STEP v0

Configure Part | Revise CAD | + Upload Drawings | Remove

Repeat Part

Quantity - 1 +

Measurement: 194.79 mm × 20.20 mm × 20.20 mm, 31688.34 mm<sup>3</sup> | 7.669 in × 0.795 in × 0.795 in, 1.934 in<sup>3</sup>

Process: CNC Machining

Material: Stainless Steel 303

Finish: Standard

Threads and Tapped Holes: 0

Inserts: 0

Precision Tolerance: +/- .005" (+/- 0.13mm)

Precision Surface Roughness: Smallest Roughness: 125uin/3.2um Ra

Inspection: Standard Inspection

**Expedite | Made in USA** \$1,973.03 ea.  
5 business days **\$1,973.03**

Standard | Made in USA \$1,170.42 ea.  
8 business days **\$1,170.42**

Economy | Made in USA \$980.72 ea.  
13 business days **\$980.72**

Economy | Made Internationally \$527.20 ea.  
12 business days **\$527.20**

#### Lead Time

Made in USA

- Expedite - 5 Business Days \$1,973.03
- Standard - 8 Business Days \$1,170.42
- Economy - 13 Business Days \$980.72

Made Internationally

- Economy - 12 Business Days \$527.20

[Learn More](#)

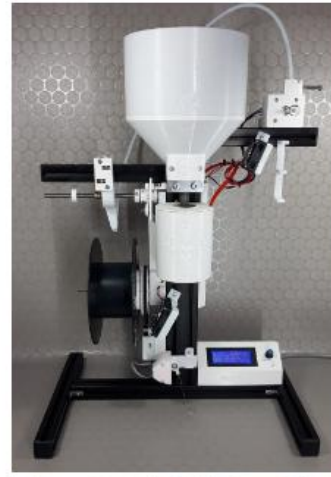
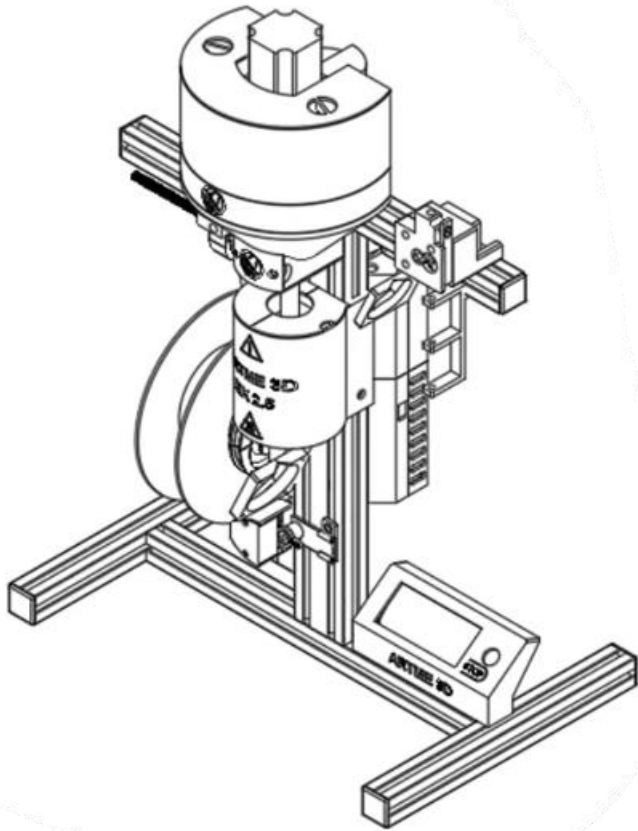
Apply Promo +

All CNC, Sheet, and 3D printing orders are eligible for free ground shipping!

Order Today to ship by August 12. Different shipping options are available at Checkout.

Subtotal

**\$1,973.03**



## Hardware DIY Kit: Original Desktop Filament Extruder MK2.5

€120,00 EUR

For information only - not available at checkout

Quantity

Add to cart

For sale here is a DIY kit of the Original Desktop Filament Extruder MK2.5 from ARTME 3D. The kit is supplied with two different extruder screws (information can be found in the item description). Please read the [product description](#) before purchasing. This is not a commercial product being sold, but a project for which the hardware is offered here in individual parts. Further items are required to set up and operate the extruder, see table.

### Delivery time:

- Please note that the preparation time before shipment is currently 10-20 days.
- [Here you will find more information on the delivery time.](#)

### Additionally required accessories and tools:

If you purchase the DIY kit from ARTME 3D, you will need the following material, which is not included in the scope of delivery, in order to be able to set up and operate the extruder:

- The 3D printed components (white parts on the photos) are not included in the scope of delivery and must be printed out yourself. [Please see the documentation for the files.](#) You can print the parts during the preparation and shipping time. It is a lot to print.
- For legal and technical reasons, the power pack is not included in the scope of delivery. Please use a closed desktop power supply (as you know it from laptops). The power pack must have 24Vdc output voltage. The output current must be at least 8 amps. In other words, have an output of at least 144 watts. It may happen that in countries with a mains voltage of 100 to 120V the power supply unit does not provide full power. Therefore, I recommend using a high-quality power supply unit for 100 to 120V mains voltage or using a more powerful power supply unit (more than 2 amperes output current). If you have the installation done by a specialist, you can also use open switching power supplies, as known from common 3D printers. However, during installation you come into contact with components that are connected to the mains voltage. Therefore, there is danger to life. Only have this type of power supply installed by a professional.

- An empty filament spool (inner hole diameter: 50 to 60mm)
- Required tool:
  - Hammer
  - Saw blade
  - Access to a vice is an advantage
  - Rolling, alcohol and paper towels
  - Needle-nosed pliers and wire strippers are useful
  - Hexkey
  - Scissors
  - Electronic caliper to measure the filament diameter (good quality)
  - Ruler

### 2 extruder screws are included:

- The extruder screw (high compression) is designed for processing stretched 3D print waste from PLA, ABS and PETG. In addition, it can also be used to process certain plastics in pellet form, such as PLA (type optimized for 3D printing) and ABS.
- The extruder screw (low compression) is designed for the



# CM-02 Extrusion-Screw

€120,00 EUR

Tax included. Shipping calculated at checkout.

Select compression

high compression ▾

Quantity

- 1 +

Add to cart





FILABOT

EX2 Filament Extruder

\$3,410.00



FILABOT

EX6 Filament Extruder - Standard Series

\$18,224.40



FILABOT

EX6 Filament Extruder - Industrial Series

\$22,890.00



FILABOT

EX6 LS Filament Extruder - Laboratory Series

\$32,712.00

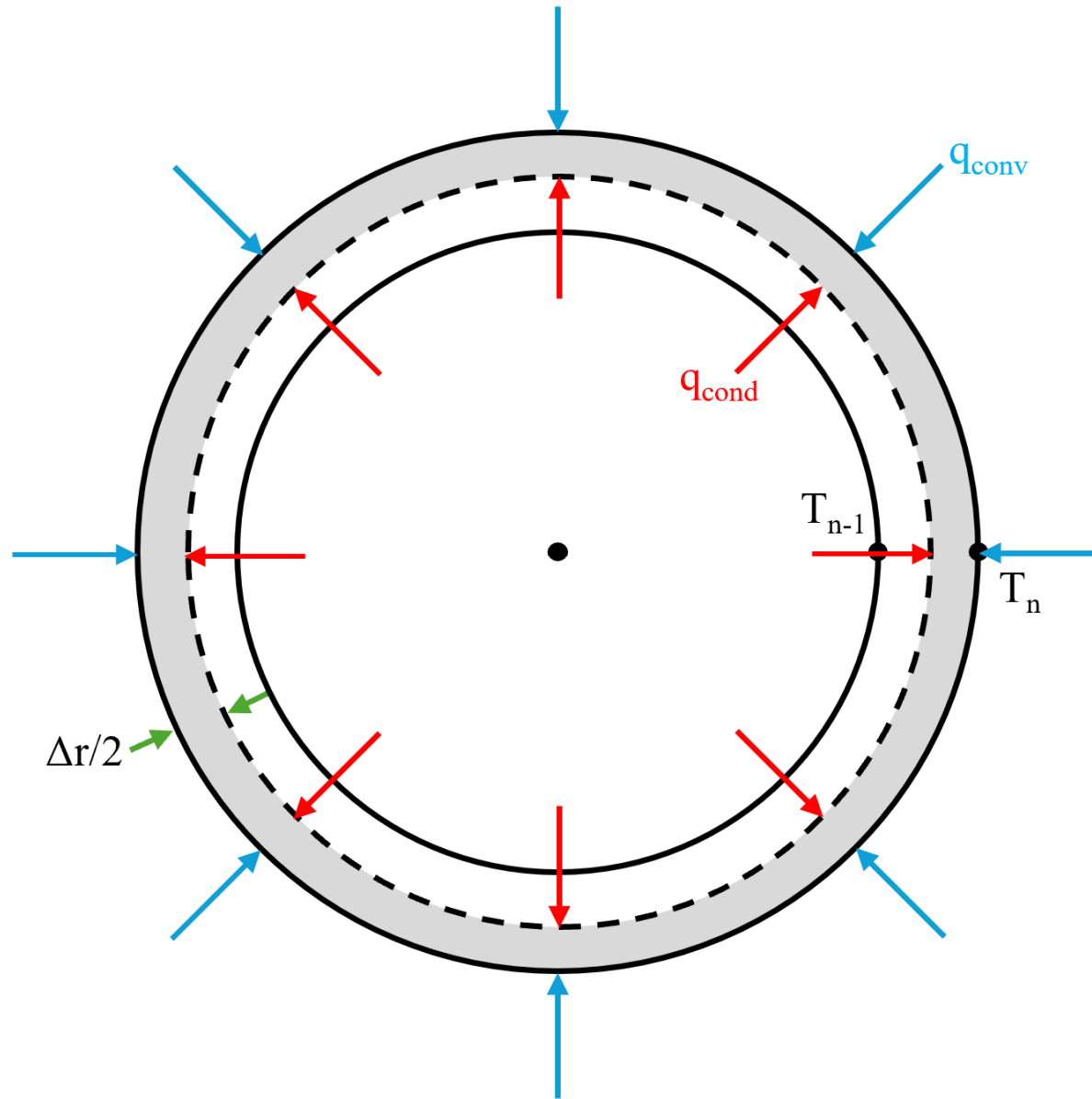


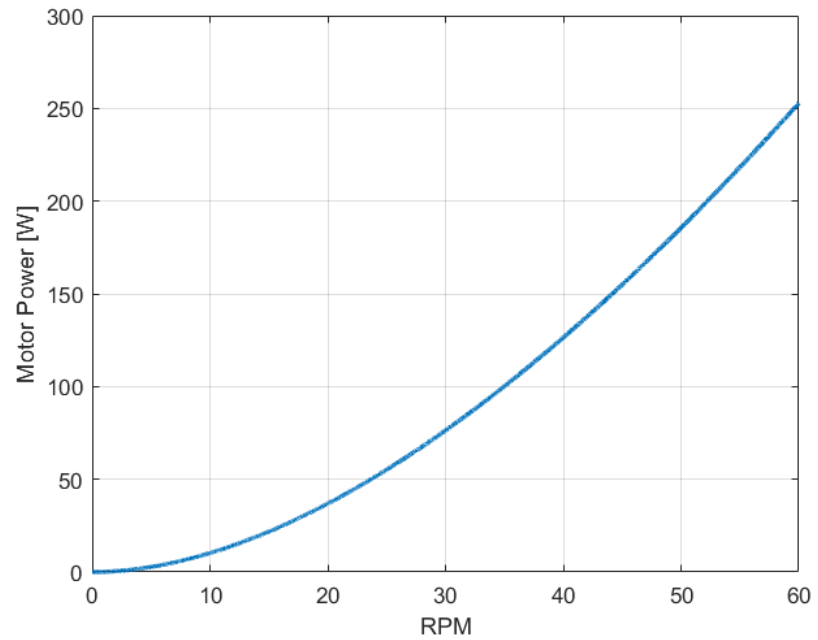
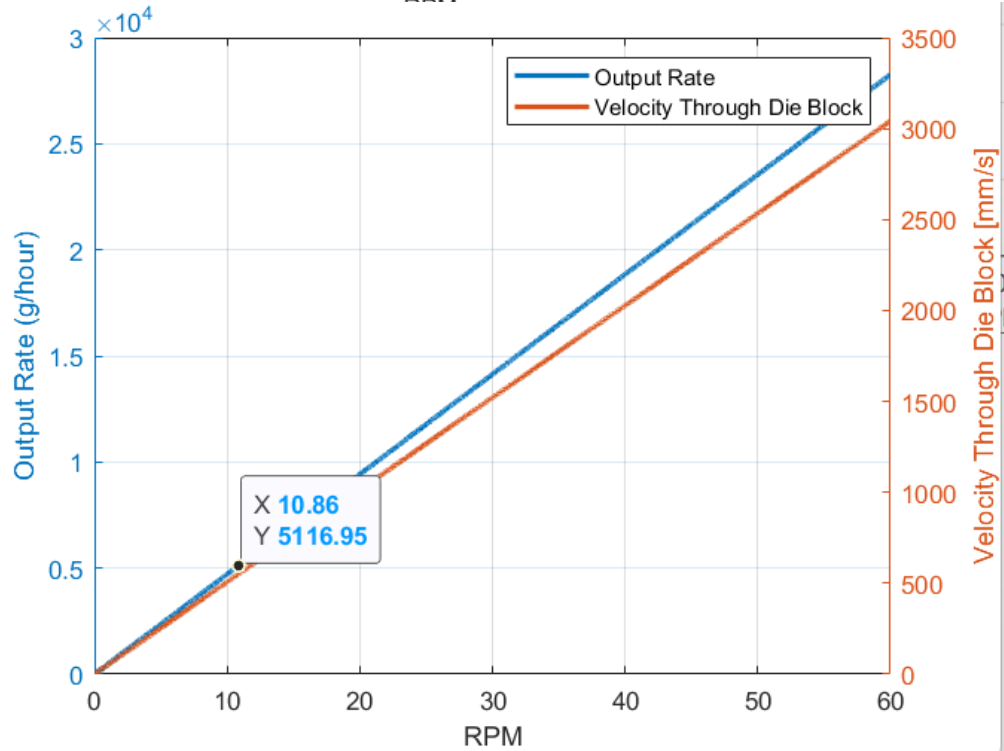
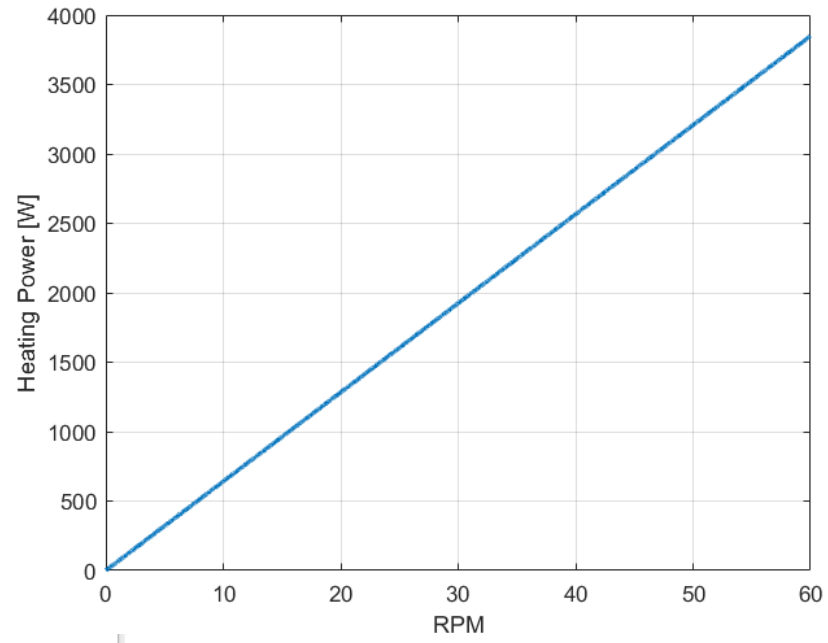
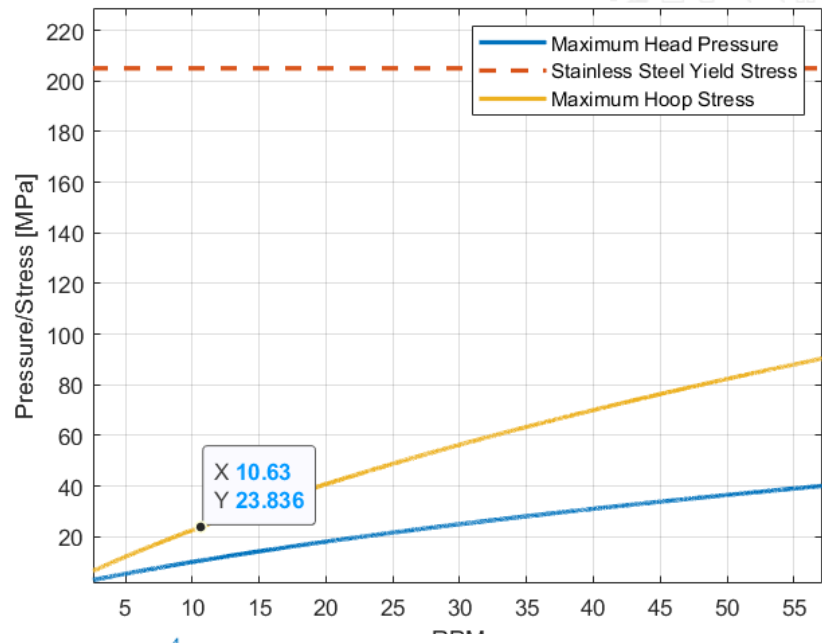
## EX6 and MDPE10 Extruder Screws

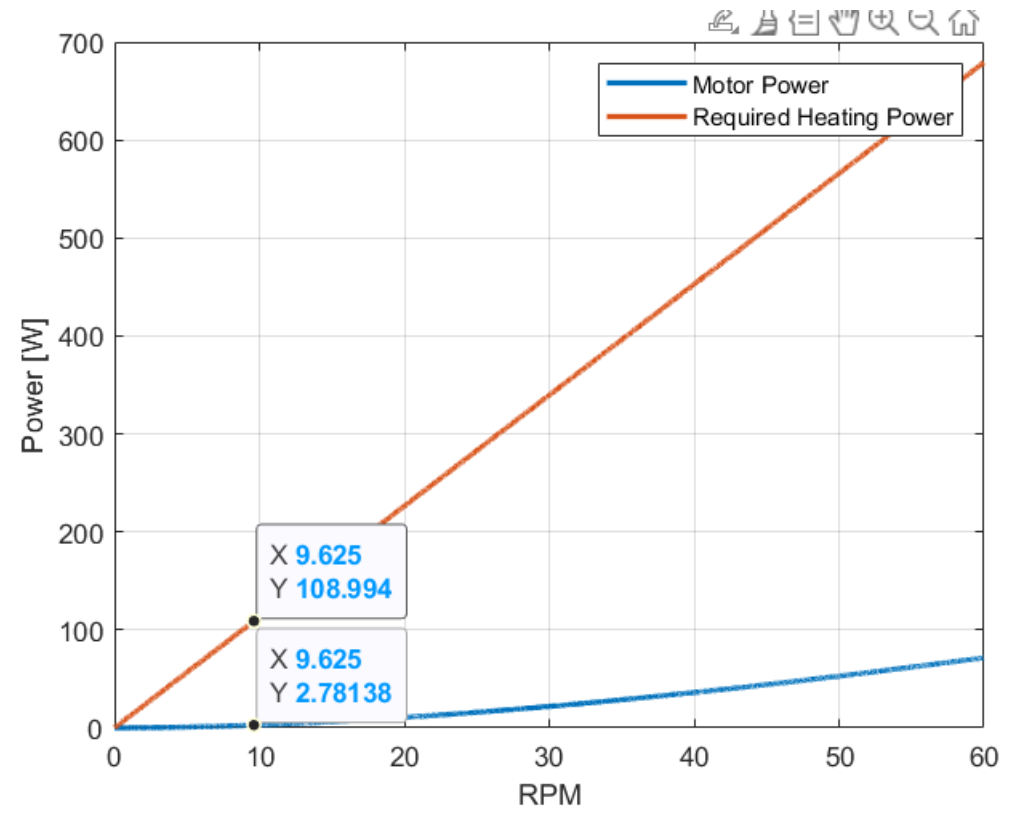
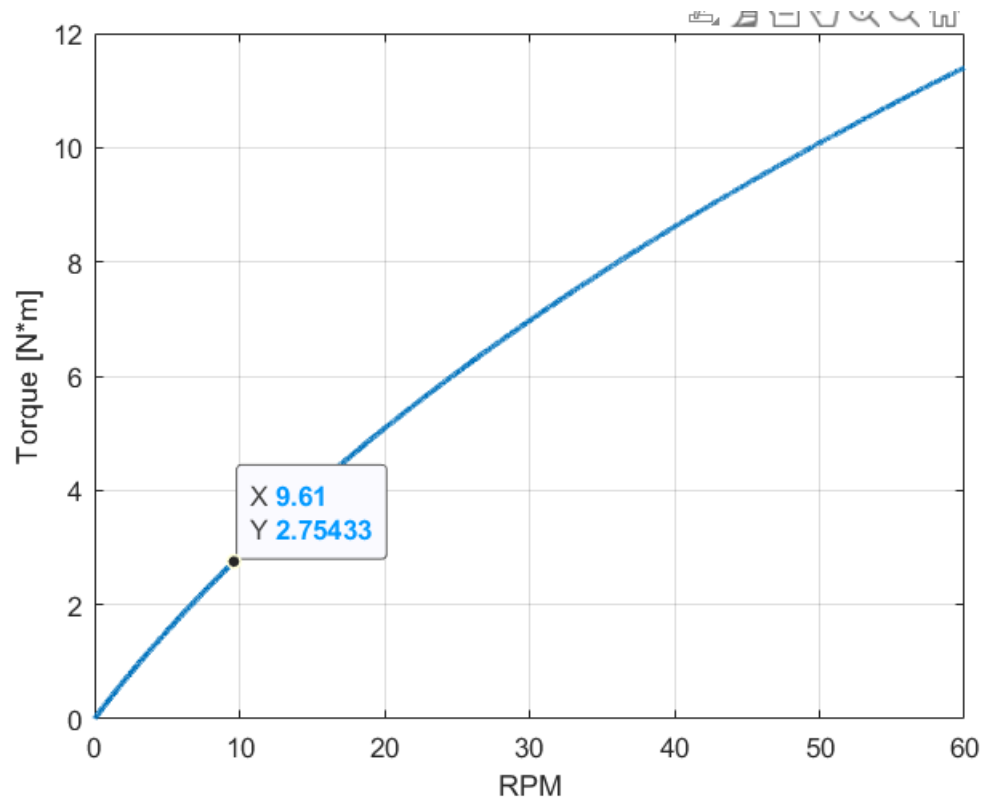
\$1,449.00

From \$130.78/mo with [shop Pay](#)

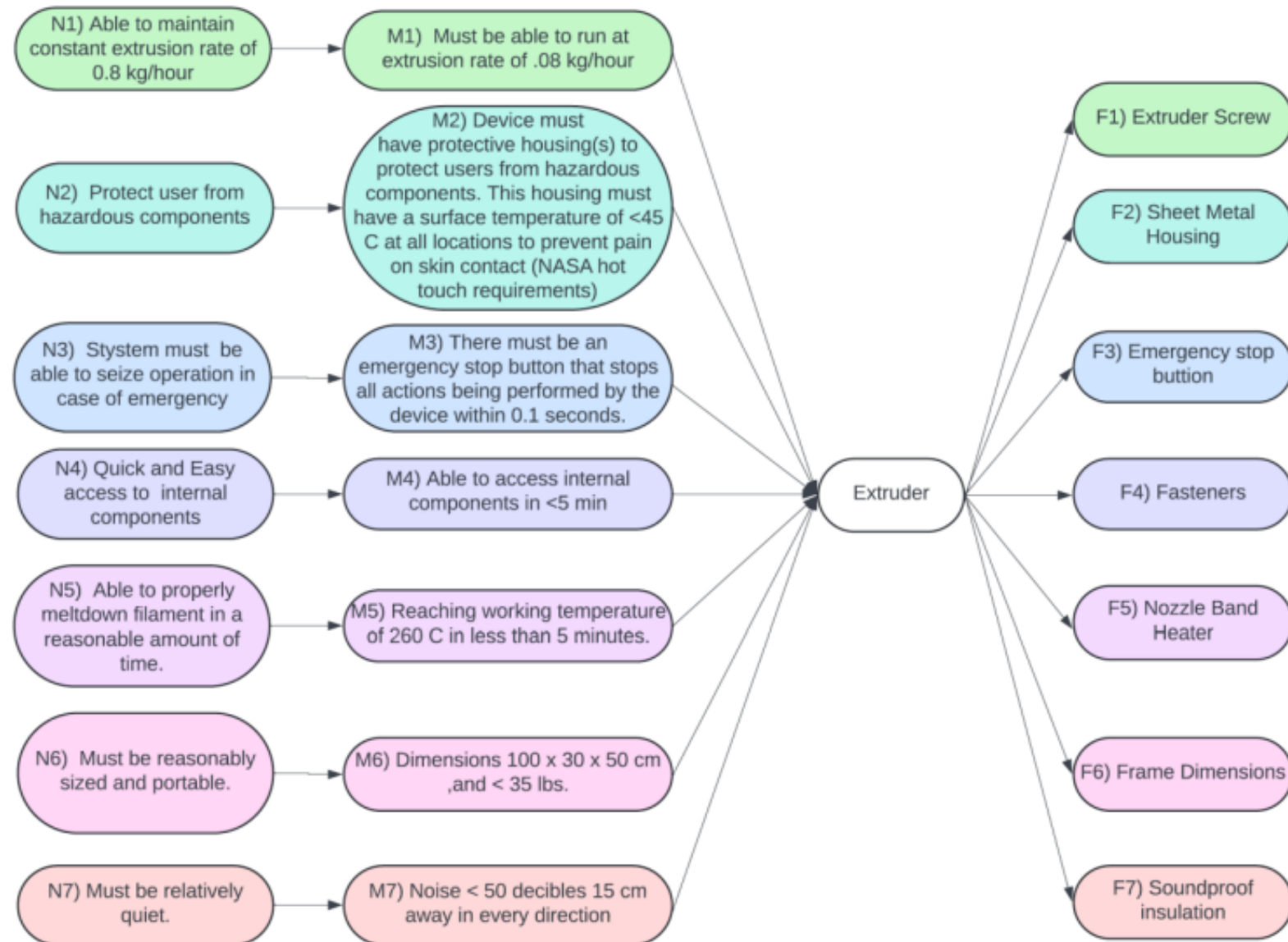




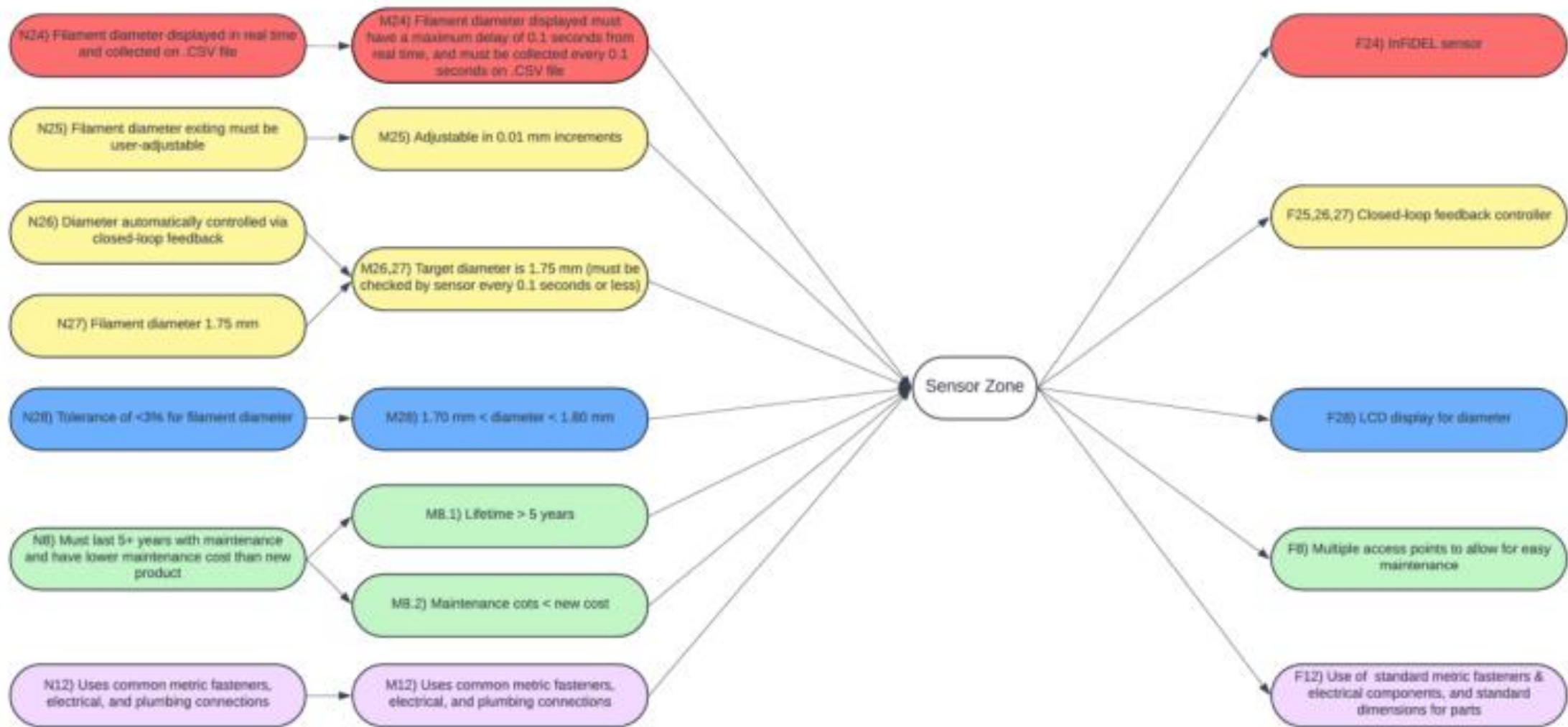




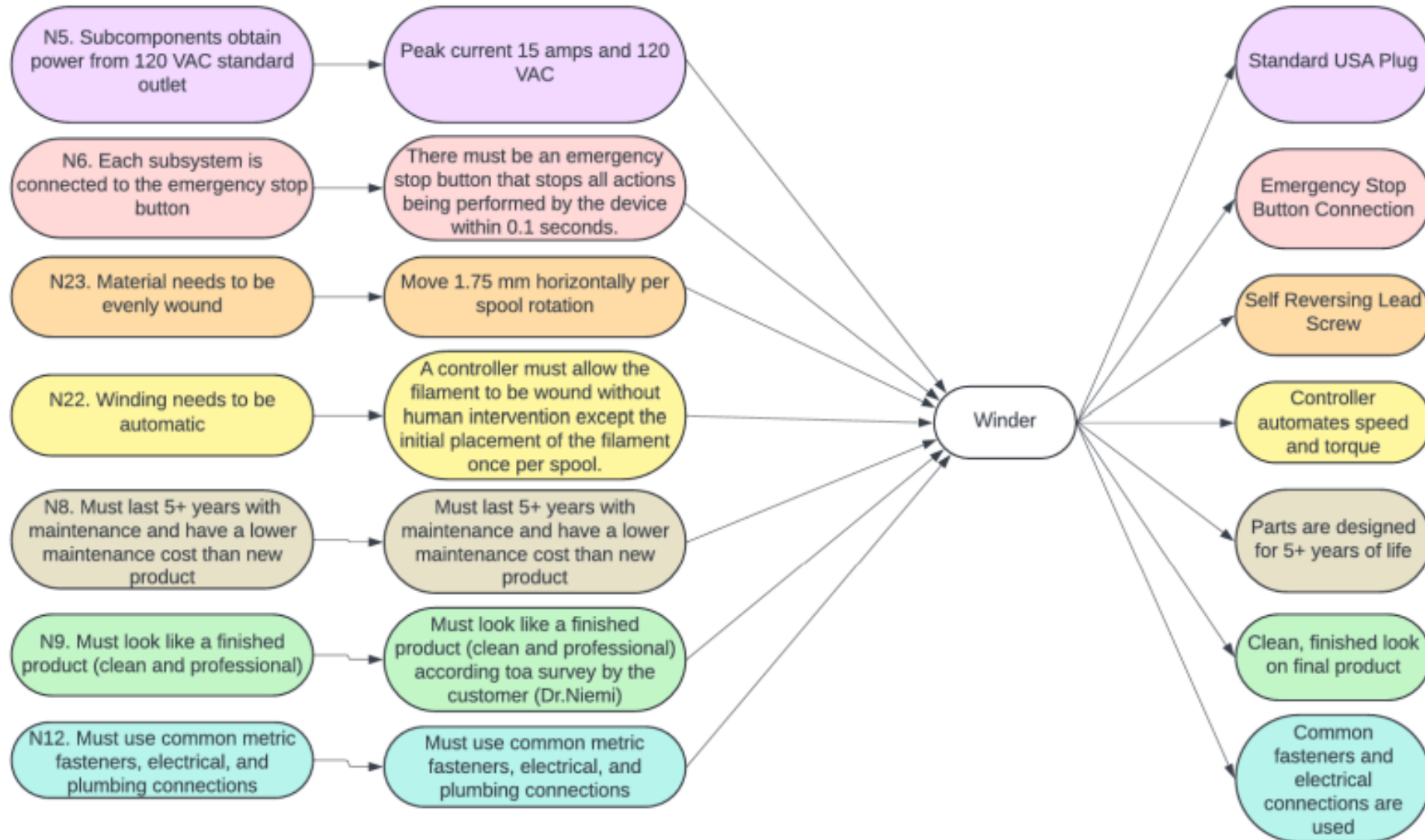
# EXTRUDER



# SENSORS



# WINDER



# OUTPUT RATE TEST STAND

